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DROPLET VAPORIZATION WITH LIQUID HEAT CONDUCTION

by

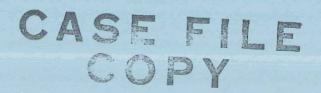
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PROPULSION SCIENCES, INC.

prepared for

NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

NASA Lewis Research Center Contract NAS 3-12030 Richard J. Priem, Project Manager



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FINAL REPORT

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by

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NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

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ABSTRACT

An analytical solution for droplet evaporation is presented. The governing differential equation includes finite liquid thermal conductivity within the droplet interior. The boundary conditions are a heat balance at the droplet surface, and finite temperatures at the droplet center. The effect of initial droplet size and temperature, droplet thermal conductivity and environmental gas pressure on temperature and vaporization rate histories are investigated. It is found that for the larger droplets, a dynamic equilibrium state is approached; i.e., the droplet surface regression rate parameter and the surface temperature approach common values.

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DROPLET VAPORIZATION WITH LIQUID HEAT CONDUCTION

By V. D. Agosta and S. S. Hammer

SUMMARY

The scope of this study is to explore the effect of liquid thermal conductivity on the behavior of an evaporating droplet. An analytical solution of the governing differential equation for heat conduction within the droplet interior is presented. The boundary conditions are finite temperature at the droplet center and a heat balance at the droplet surface. The effect of initial droplet size and temperature, droplet thermal conductivity and environmental gas pressure on temperature and evaporation rate histories are investigated.

It is found that for larger droplets, a common dynamic equilibrium state asymptotically is approached. The effect of thermal conductivity is to vary the time required for the droplet surface temperature to reach the asymptotic value. The time to vaporize ten percent of the initial droplet mass was not influenced by the thermal conductivity. The droplet equilibrium surface regression rate and temperature increases with environmental gas pressure. The droplet surface temperature reaches the critical value at an environmental gas pressure that is dependent on the volatility of the fluid. Finally, for droplets equal or greater than 100 microns diameter and with realistic thermal conductivities, the temperature gradients attained at 10^{-4} sec. are contained in the outer ten percent of the droplet radius.

INTRODUCTION

The burning rate for small fuel droplets is usually based on the quasi-steady envelope flame model (Refs. 1-4). Essentially, this model assumes that heat is transferred back from the flame envelope such that vaporization is sustained, and that the burning rate is equal to the flow of fuel from the interior of a droplet through a sphere of the same size and located in the same environment. It is further assumed that either the temperature at the droplet surface is near the boiling point or is uniform throughout the droplet. The quasi-steady droplet evaporation model appears to break down near the propellant critical point (Ref. 5).

The primary objective of the analysis presented herein is to determine the behavior of vaporizing droplets in which heat conduction in the liquid droplet is included in the model. This is accomplished by analyzing the droplet interior and employing the environment as boundary conditions. Since the droplet is the mass source, then it is argued that its behavior must be analyzed subject to the constraints of its environment. An additional bonus that becomes apparent is that the model presented herein can serve as one for the evaporation behavior of droplets near the droplet critical conditions.

It is our pleasure to acknowledge Dr. Gino Moretti, who developed the computer program for this study.

THEORY

Initially, a droplet with uniform temperature is introduced into a hot atmosphere which is essentially at the adiabatic flame temperature. Droplet heating and vaporization occur simultaneously from the initial conditions and approach asymptotically a dynamic equilibrium state. The droplet vapors diffuse, mix and react chemically with the hot gases. Thus the physical structure of the model consists of a spherical droplet surrounded by a film of reacting vapor-gas mixture. The thickness of the film is defined as that distance where the reaction is in equilibrium for the local or given $^{\rm O}/{\rm F}$ ratio. Heat transfer, ${\rm q}_{\rm T}$, is assumed to occur backward toward the droplet which goes: (1) toward heating up the diffusing vapors, ${\rm q}_{\rm Sh}$, (2) supplying the heat of vaporization, ${\rm q}_{\lambda}$, and (3) heating up the droplet, ${\rm q}_{\rho}$, Fig. 1.

The governing differential equation for heat conduction in a spherical liquid droplet is given by

$$T_{t} = \alpha (T_{rr} + \frac{2}{r} T_{r}) \qquad 0 \le r \le r_{s}$$

$$t \ge 0$$
(1)

where

T is the temperature,

r radius, r_s surface radius,

t time, and

 α liquid thermal diffusivity.

The subscripts r and t represent differentiation with respect to

radius and time, respectively. For small droplets the assumption of spherical geometry is usually accepted. It is recognized that where drag exists, droplets deform; however, in this analysis the deformation is neglected. The initial condition assumes that the droplet is at uniform temperature $\mathbf{T}_{\mathbf{0}}$,

$$T(r,0) = T_0$$
 $0 \le r \le r_s$ (2)
 $t = 0$

The boundary conditions are

$$T(0,t)$$
 is finite $r = 0$ (3)
 $t \ge 0$

and

$$h(T_{f}^{-T}s) = \mu T_{r}^{+} \frac{\dot{w}}{A_{d}} [\lambda + (C_{p})_{v} (T_{f}^{-T}s)]$$

$$r = r_{s}$$

$$0 < t < \infty$$
(4)

where

is the heat transfer coefficient,

Tf ambient (adiabatic flame) temperature,

droplet surface temperature,

thermal conductivity of droplet,

heat of vaporization,

(cp) vapor specific heat,

w mass evporation rate,

Ad droplet surface area.

It is assumed that the film thickness surrounding the droplet is small compared to the droplet diameter.

The problem is still not completely defined because the mass evaporation rate is unknown. Thus, a constitutive equation is required and is

$$\rho_{\ell}\dot{r}_{s} = \frac{\dot{w}}{A_{d}} = K_{g}p \ln \frac{p}{p-p_{v}}, r = r_{s}$$
 (5)

where

r is the droplet surface regression rate,

the liquid density,

K mass transfer coefficient,

p⁹ total gas pressure,

vapor pressure corresponding to the droplet temperature
surface.

It is assumed that the Clausius-Clapeyron equation relates the droplet surface temperature to the vapor pressure. A general relationship for this purpose is

$$lnp_{V} = a + \frac{b}{T} + cT$$
 (6)

where a, b, and c are constants to be determined from thermodynamic data. The above Eq. (6) is also used for interpolation purposes. The heat of vaporization is a function of the droplet surface temperature and is given by

$$\lambda \left[1 - \frac{T_{s}}{T_{c}} \right]^{\gamma} \tag{7}$$

wjere λ and γ are constants and T the critical temperature. The average vapor specific heat, $\overline{(c_p)}_V$, is either given by suitable equations, e.g.,

$$\left(\overline{c_{p}}\right)_{v} = \frac{\int_{T_{s}}^{T_{f}} \left(\overline{c_{p}}\right)_{v} dT}{T_{f} - T_{s}}$$
(8)

or from tables. The heat and mass transfer coefficients h and $\mathbf{K}_{\mathbf{q}}\text{,}$ respectively, are obtained from the Nusselt correlations,

$$\frac{2r_{s}^{h}}{K_{m}} = 2 + 0.6(Pr)^{1/3} (Re)^{1/2}$$
 (9)

and

$$\frac{2r_{s}^{R} _{o}^{TK}_{q}}{M_{y}^{D}} = 2 + 0.6(Sc)^{1/3} (Re)^{1/2}$$
 (10)

where

is the vapor-gas mixture thermal conductivity, K_{m} universal gas constant, vapor molecular weight, molecular diffusion coefficient, arithmetic mean temperature, $\frac{T_{f+}T_{s}}{2}$ D Prandtl number $(c_p \mu/n)_m$ PrSchmidt number $(\mu/D_{\rho})_{m}$. Sc Reynolds number $2r_s(u-v)(\frac{p}{u})_m$, Re vapor-gas mixture viscosity, μ vapor-gas mixture density, gas velocity, u

It is recognized that most of the thermodynamic properties are functions of droplet surface temperature. However, to include such functional relationships at this time is impractical from a computational point-of-view.

droplet velocity.

A Laplace transform procedure was used to obtain a solution for the temperature profile in the liquid droplet employing the additional assumption that

$$\dot{r}_{s}\Delta t \ll r_{s}$$
 (11)

The solution is given below (see Appendix A for derivation):

$$T=T_{o}-\left[\frac{T_{o}(1+Ar_{s})+B}{Ar}\right]\cdot\left\{erfc\frac{1-R}{2\sqrt{\tau}}-erfc\frac{1+R}{2\sqrt{\tau}}-erfc\frac{1+R}{2\sqrt{\tau}}-erfc\frac{1-R}{2\sqrt{\tau}}\right\}$$

$$exp(Ar_{s}(1-R)+A^{2}r_{s}^{2}\tau)\cdot erfc(\frac{1-R}{2\sqrt{\tau}}+Ar_{s}\sqrt{\tau})+erfc(\frac{1+R}{2\sqrt{\tau}}+Ar_{s}\sqrt{\tau})\right\}$$

$$exp(Ar_{s}(1+R)+A^{2}r_{s}^{2}\tau)\cdot erfc(\frac{1+R}{2\sqrt{\tau}}+Ar_{s}\sqrt{\tau})\right\}$$

$$(12)$$

where

$$R = \frac{r}{r_s}$$

$$\tau = \frac{\alpha t}{r_s^2}$$

$$A = \left[\frac{h}{\kappa} - \frac{1}{r_s} - \frac{(\overline{c_p})_v}{\kappa} K_g p \ln \frac{p}{p - p_v}\right]$$

and

$$B=r_{s}[(\frac{\lambda}{n} + \frac{(\overline{c_{p}})_{v}T_{f}}{n})K_{g}p \ln \frac{p}{p-p_{v}} - \frac{hT_{f}}{n}].$$

The analysis of droplet evaporation in a gas stream, as formulated, is developed into a computer program. Once the initial conditions are specified at time t=0 (e.g., the initial droplet radius, the initial uniform droplet temperature and the ambient conditions), one obtains a formula for the surface temperature of the droplet at any successive time, t₁. A number of coefficients in that equation depend on mean values of physical parameters over the entire span t=0,t₁ (physical properties used in this study are given in Appendix B). Therefore, the computation has to be performed stepwise, in order to provide averaged intermediate values of the physical parameters between t=0 and t=t₁ (see Appendix C for calculation procedure and computer program).

The program is as follows: Time is increased by steps. At each step, the surface temperature and the droplet radius are averaged from the initial values to the values pertinent to that step. The mean physical properties are computed, according to

the formulae given. Then the new surface temperature is computed from the Laplace transform solution and the rate of decrease of droplet radius is evaluated. This allows the new droplet radius at the next step to be determined. Since some of the yet unknown new values have to be used in making the averages, one has to start each step with a suitable guess on the surface temperature and then find the right value by a trial-and-error technique.

The program is written in such a way that different gases and different liquids can be considered, by changing some of the data cards.

A computer run is terminated when ten percent of the initial droplet mass is evaporated, or when the droplet surface temperature reaches the propellant critical temperature. The regression rates and times are normalized by using steady state droplet evaporation rates and lifetimes. It is assumed a priori that for various droplet diameters, the droplet surface temperatures approach a common value during evaporation (i.e., a wet-bulb temperature) (Ref. 6). Thus, considering the equation for mass transfer from the droplet surface, Eq. (5),

$$\frac{\dot{w}}{A_d} = \dot{r}_{s} \rho_{\ell} = K_g p \ln \frac{p}{p - p_v}$$
 (5)

the chamber pressure, p, and the liquid density, ρ_{χ} , are given constants. The propellant vapor pressure, p_{χ} , is constant because of the aforementioned assumption. Therefore, Eq. (5) reduces to

$$\dot{r}_{s} = c_{1}K_{g} \tag{13}$$

where C_1 is determined from the constant terms in Eq. (5). Nusselt's relation for mass transfer, Eq. (10) is given as

$$\frac{2r_s R_o^T K_g}{M_V^D} = 2 + 0.6 (sc)^{1/3} (Re)^{1/2}$$
 (10)

where r is the droplet radius.

Assuming that for high speed flow,

$$0.6(Sc)^{1/2}(Re)^{1/2} > 2$$
 (14)

gives

$$K_g \propto r_s^{-1/2}$$
 (15)

Combining Eqs. (13) and (15) gives

$$r_{s}r_{s}^{1/2} \approx constant$$
 (16)

If the decrease in droplet radius during the initial stages of evaporation is neglected,

$$r_{s} = r_{0} \tag{17}$$

Thus,

$$r_{s}^{1/2} = constant , (18)$$

where r is the initial droplet radius.

This variable is used as the normalized surface regression rate. Integrating Eq. (16) between the limits 0 \leq r $_{\rm S}$ \leq r $_{\rm O}$ and 0 \leq t \leq t $_{\rm d}$, where t $_{\rm d}$ is the droplet lifetime, gives

$$t_{d} \propto r_{o}^{3/2} \tag{19}$$

Thus, the nondimensional time is defined as

$$\frac{t}{t_d} = C_2 \frac{t}{r_0^{3/2}}.$$
 (20)

RESULTS

The results for droplet evaporation for four propellant combinations are given. The parameters varied are: the initial droplet radius, the initial droplet temperature, the thermal conductivity, and the chamber pressure. The reference case for all propellants is an initial droplet diameter of 100 microns, initial temperature of the normal boiling point, and the chamber pressure equal to that of the evaporating propellant critical pressure.

A. Variation in Droplet Histories with Initial Droplet Radius

__In Fig. 2, the droplet surface regression rate parameter $\dot{r}\sqrt{r}$ is plotted against the nondimensional time for the following propellant combinations:

Fig. 2a 50/50 aerozine with NTO

Fig. 2b NTO with 50/50 aerozine

Fig. 2c oxygen with gaseous hydrogen

Fig. 2d n-heptane with gaseous oxygen.

The propellants are at their normal boiling points and the ambient pressure is that of the propellant critical pressure.

Several general statements can be made for all these propellant combinations: (1) The surface regression rate parameter curves for all the droplet sizes do not coalesce during the heating up period, indicating different histories; (2) As time increases, the surface regression rate parameters approach a common value for the larger droplets. The surface regression rates for the smaller droplets (e.g., 25µ droplet in particular) continue to increase, indicating that "steady state" behavior is not approached; (3) It is noticed that the curves for the surface regression rate parameter cross, and that the value of the non-dimensional time at which crossover occurs decreases as the propellant becomes more volatile. This result suggests that the heating up period decreases with propellant volatility; (4) As the propellant becomes more volatile, the asymptotic value of the burning rate parameter increases.

In Fig. 3, the droplet surface temperature history is plotted against the nondimensional time. It is observed that in all cases,

a common surface temperature is approached asymptotically for each propellant. The reduced surface temperature, i.e., $T_{\rm s}/T_{\rm c}$, where $T_{\rm c}$ is the critical temperature of the droplet propellant, varies from 0.77 for n-heptane to 0.97 for nitrogen tetroxide. From these results, it is seen that the reduced surface temperature increases as the volatility of the propellant becomes greater.

Fig. 4 shows the droplet temperature distribution with radius at 10^{-4} sec. We observe two trends, namely: (1) for each propellant and for all droplet sizes, the surface temperatures approach a common value; and (2) steep positive temperature gradients occur in approximately the outer ten percent of the droplet radius. In one case, i.e., the 25 micron diameter droplet of n-heptane, the temperature gradient is broadened and extends to 30% of the droplet radius. However, if instead of plotting the droplet temperature distribution at the end of 10^{-4} sec., the nondimensional time at the crossover point is used, then for the case of the 25 micron droplet of n-heptane, the surface temperature gradient is contained in the outer ten percent of the droplet radius.

B. Variation in Droplet Histories with Initial Droplet Temperature

In the following series of computer tests, the initial droplet temperature was varied. In these cases the initial droplet diameter was assumed to be 100 microns, and the gas pressure that of the droplet propellant critical pressure. Three runs were made with the droplet initially at: (1) the propellant normal boiling point, (2) the temperature $[T_{\rm nbp} + 5/6T_{\rm c}]/2$, and (3) the temperature $5/6T_{\rm c}$. The results of these tests are plotted in Figs. 5, 6 and 7. In Fig. 5, the surface regression rate parameter is plotted against the reduced time. It is seen that with 50/50 aerozine and n-heptane, the surface regression rate parameter approaches an asymptotic value independent of initial propellant temperature. With NTO and oxygen, the curves do not appear to cross within the real time of 10^{-4} sec. Certainly the crossover point, if any, has translated to larger values of time. behavior is observed for the surface temperatures versus time, Fig. 6; namely, it is seen that a common value of surface temperature is approached asymptotically for each propellant.

In Fig. 7, the temperature distribution within the droplet is shown for the time 10^{-4} sec. Again it is seen that the surface temperatures approach a common value for each propellant, and that the temperature gradient is located in the outer ten percent of the droplet radius. One significant departure is observed for n-heptane with an initial temperature of $810^{\circ}R$. The temperature gradient at the surface is negative. In this case the initial droplet temperature was above the equilibrium surface temperature. The dynamic balance of heat and mass transfer, Eq. (4), requires that part of the droplet internal energy be used to supply the heat of vaporization.

C. Variation in Droplet Histories with Droplet Thermal Conductivity

In the following computer test runs, the effect of liquid thermal conductivity on the droplet evaporation behavior and temperature distribution was considered. The results of this study are plotted in Figs. 8, 9 and 10. In all these tests, the data extends to where ten percent of the droplet mass has evaporated. The liquid thermal conductivity was varied between 10^{-5} and 10^{-1} BTU ft. $^{-1}$ sec $^{-1}$ or $^{-1}$ c. Values lower than 10^{-5} or greater than 10^{-1} caused computational procedure errors due to the form of the equations.

In Fig. 8, the surface regression rate parameter is plotted versus the nondimensional time. For 50/50 Aerozine and n-heptane, steady state evaporation is not approached for the cases where the larger thermal conductivities are used. This observation is based on the fact that the latter curves do not level off and approach common asymptotic values of the surface regression rate by the time ten percent of the mass is vaporized. The more volatile propellants are less sensitive to the value of thermal conductivity and level off toward common values of regression rate, thus attaining equilibrium values sooner. For all cases it is noticed that for smaller values of thermal conductivity, the surface regression rate parameter increases faster. general remarks can be made based on the surface temperature data given in Fig. 9. In Fig. 10, the droplet temperature distribution is shown. The results agree with intuition in that the greater the liquid thermal conductivity, the more uniform is the droplet temperature.

In passing it is noted that the time required for a 100 micron droplet to reach the ninety percent mass point was about the same, i.e., $5 \times 10^{-5} \le t_{90\%} \le 10^{-4}$ sec., regardless of the thermal conductivity. This behavior is due to the fact that only a small fraction of heat transfer, q_T , goes toward heating up the droplet, q_ℓ . However, the behavior of these heated droplets in a time dependent environment could be vastly different.

D. Variation in Droplet Histories with Environment Gas Pressure

In this series of computer tests, the environmental gas pressure was increased from the propellant critical pressure to that where the droplet surface temperature reached the propellant critical temperature. In the data no evidence is seen of an approach to an asymptotic value of surface temperature. This behavior appears to be in agreement with that observed by Faeth (Ref. 7).

In Fig. 11, the droplet surface temperatures are plotted versus the reduced gas pressure, p_r , for 100 micron droplets initially at normal boiling point temperature. As the pressure increases, the surface temperature increases. The values plotted represent the droplet surface temperature at a time when ten percent of the droplet mass has evaporated. For this ninety percent mass point, the reduced pressure at which the critical temperature is reached at the surface is:

It is observed that as the propellant volatility increases, the reduced pressure decreases.

In Fig. 12, the surface regression rate is shown as a function of chamber pressure. It is seen that the surface regression rate increases as the chamber pressure increases. The trend, therefore, is for the droplet lifetime to decrease as the chamber pressure increases. This is in agreement with the work of Faeth (Ref. 7).

SUMMARY OF RESULTS

The evaporative behavior of a liquid propellant droplet has been determined from a solution to a system of equations which include heat transfer within the droplet. In general, for droplets larger than 100 microns in diameter, the droplet heats up within 10^{-4} sec and approaches a steady state of evaporation. Smaller droplets, e.g., 25 micron diameter droplets, do not approach a steady state behavior. The evaporative behavior of droplets with diameter larger than 100 microns is correlated by the surface regression rate parameter, $\dot{\mathbf{r}}_{\rm S} \sqrt{\mathbf{r}_{\rm O}}$ and the nondimensional time, t/t_b after steady state have been achieved.

Analytical results obtained from the solution are:

- (1) Regardless of the droplet initial temperature or radius, a common value of the droplet surface temperature is asymptotically approached. Therefore, where the droplet is heat transfer limited, or where the droplet initial temperature is greater than the steady state surface temperature, the droplet internal energy can contribute to the heat of vaporization.
- (2) The effect of thermal conductivity is to vary the time required for the droplet surface temperature to reach the asymptotic value. The time required to evaporate ten percent of the droplet mass was not influenced by the value of the liquid thermal conductivity.
- (3) The droplet equilibrium surface regression rate and temperature increases with environmental gas pressure. The droplet surface temperature reaches the critical value at an environmental gas pressure that is dependent on the volatility of the fluid.
- (4) For droplets equal to or greater than 100 microns diameter and with realistic thermal conductivities, the temperature gradients attained at 10^{-4} sec are contained in the outer ten percent of the droplet radius.

APPENDIX A

ANALYTIC SOLUTION FOR TEMPERATURE DISTRIBUTION

A short time solution for the temperature distribution in a liquid sphere where evaporation occurs at the surface is obtained by utilizing Laplace transformations. Eq. (1) can be transformed to a more convenient form by the substitution

$$U = Tr (A-1)$$

which gives

$$U_{t} = \alpha U_{rr} \qquad 0 \le r \le r_{s}$$

$$t > 0$$
(A-2)

The initial condition expressed in Eq. (2) is

$$U(r,0) = rT_{O} \qquad 0 \le r \le r_{S}$$

$$t=0 \qquad (A-3)$$

and the boundary conditions corresponding to Eqs. (3a) and (3b) are

$$U(0,t) = 0$$
 $r=0$ $t \ge 0$ (A-4a)

$$U_{r} = U\left[\frac{1}{r_{s}} - \frac{h}{n} + \frac{\overline{(c_{p})}_{v}}{n} \dot{r}_{s}\rho_{\ell}\right] - \dot{r}_{s}\rho_{\ell} \frac{r_{s}}{n} [\lambda + \overline{(c_{p})}_{v} T_{f}] + \frac{r_{s}h}{n} T_{f}$$

It is assumed that $\overset{\bullet}{r}_{s}\Delta t << r_{s},$ and that λ and p_{v} are constants over the time interval $\Delta t.$

The Laplace transform is taken of Eq. (A-2),

$$\int_{0}^{\infty} e^{-pt} u_{rr} dt - \frac{1}{\alpha} \int_{0}^{\infty} e^{-pt} u_{t} dt = 0$$
 (A-5a)

Expanding and including the initial condition of Eq. (A-3) gives

$$\overline{U}_{rr} - q^2 \overline{U} = -\frac{r}{\alpha} T_0$$
 (A-5b)

where
$$q^2 = \frac{p}{\alpha}$$
. (A-5c)

The boundary conditions of Eqs. (A-4a) and (A-4b) becomes

$$\overline{U}_{r} + A\overline{U} + \frac{B}{p} \neq 0 \qquad r = r_{s}$$

$$t \geq 0$$
(A-6b)

where

$$A = \left[\frac{h}{n} - \frac{1}{r_s} - \frac{(\overline{c_p})_v K_g p}{n} \ln \frac{p}{p - p_v}\right]$$
 (A-7a)

and

$$B = r_s \left[\left(\frac{\lambda}{\kappa} + \frac{\left(c_p \right)_v r_f}{\kappa} \right) K_g p \ln \frac{p}{p - p_v} - \frac{h r_f}{\kappa} \right]$$
 (A-7b)

The general solution to Eq. (A-5b) is

$$\overline{U} = c_1 e^{qr} + c_2 e^{-qr} + \frac{T_0 r}{q^2 \alpha}$$
 (A-8)

Substitution into boundary condition (A-6a) gives

$$C_2 = -C_1 \tag{A-9a}$$

and into Eq. (A-6b) gives

$$c_{1} = -\frac{\frac{T_{0}}{2}[1+Ar_{s}] + \frac{B}{p}}{(A+q)e^{qr_{s}} - (B-q)e^{-qr_{s}}}$$
(A-9b)

Thus the solution becomes

$$\frac{T}{U} = \frac{T_{O}r}{p} - \left[\frac{T_{O}(Ar_{S}+1)+B}{p}\right] \cdot \left[\frac{C}{(q-A)} + \frac{C}{(q+B)} + \frac{C}{(q+B)}$$

For a solution valid over a short interval of time, the series can be truncated after the n=0 term. This was verified by extending the series and testing for convergence. The inverse transform of Eq. (A-10) is

$$T = T_{O} - \left[\frac{T_{O}(1+Ar_{S})+B}{Ar}\right] \cdot \left\{ erfc \frac{1-R}{2\sqrt{T}} - erfc \frac{1+R}{2\sqrt{T}} - erfc \frac{1+R}{2\sqrt{T}} - erfc \frac{1-R}{2\sqrt{T}} - erfc \frac{1-R}{2\sqrt{T}} + Ar_{S}\sqrt{T}\right) + exp(Ar_{S}(1+R)+A^{2}r_{S}^{2}\tau) \cdot erfc \left(\frac{1-R}{2\sqrt{T}} + Ar_{S}\sqrt{T}\right) \right\}$$

$$= \exp(Ar_{S}(1+R)+A^{2}r_{S}^{2}\tau) \cdot erfc \left(\frac{1+R}{2\sqrt{T}} + Ar_{S}\sqrt{T}\right) \right\}$$
(A-11)

A qualitative discussion of the solution is now given. If we refer to Eq. (A-ll), we find that it is made up of two parts. The first is, $T_{\rm O}$, and it is recognized as the droplet initial temperature. The second is the product of

$$\left[\frac{T_{o}(1+Ar_{s})+B}{Ar}\right] \cdot F(r,t)$$

and is the droplet heatup and vaporization due to the backward heat transfer from the hot environment, i.e., for the case where the environment is considered steady in temperature. Therefore, the two parts make up the solution to the heat and mass transfer

to a burning droplet in a constant temperature environment. From summary slide rule calculations it is found that for hydrazine-type and hydrocarbon-type fuels, droplet heatup times are about 10^{-4} sec. In addition, steep temperature gradients occur at the droplet surface, the interior apparently not affected in the absence of radiation and convection. Thus the transient behavior of such a droplet is different from one with infinite thermal conductivity (i.e., a constant temperature droplet) due to decreased thermal inertia.

APPENDIX B

THERMODYNAMIC CONSTITUTIVE EQUATIONS

The thermodynamic properties for the propellant combinations employed in this study are given below. Where mixtures are considered, then Gibbs-Dalton's Law is used:

pressure,
$$p = \sum_{i} p_{i}$$
 (B-1)

enthalpy,
$$h = \sum_{i} y_{i}h_{i}$$
 (B-2)

entropy,
$$s = \sum_{i} y_{i} s_{i}$$
 (B-3)

where y is the mass fraction, and for transport properties,

$$\theta = \sum_{i} \frac{x_{i}\theta_{i}}{\sum_{i} x_{i}\phi_{i,i}}$$
 (B-4)

where

$$\varphi_{ij} = \frac{1}{\sqrt{8}} \left(1 + \frac{M_i}{M_j}\right)^{-\frac{1}{2}} \left[1 + \left(\frac{\theta_i}{\theta_j}\right)^{\frac{1}{2}} \left(\frac{M_i}{M_j}\right)^{\frac{1}{2}}\right]^2$$
 (B-5)

and θ is the transport property,

M the molecular weight,

x the mole fraction.

For the propellant combinations, 50/50 Aerozine with nitrogen tetroxide, oxygen with gaseous hydrogen, and nitrogen tetroxide with 50/50 Aerozine, the following constitutive equations are used:

Diffusion coefficient equation,

$$D_{r} = \frac{2.33 \times 10^{-6} (P_{CA} P_{CB})^{1/3}}{\sqrt{\frac{M_{A} M_{B}}{M_{A} + M_{B}}}} \cdot \frac{(T_{CA} T_{CB})^{0.416}}{p}$$

$$(\frac{T}{\sqrt{T_{CA} T_{CB}}})^{1.823}$$
(B-6)

Vapor pressure - Temperature relationship,

$$p_{V} = \exp(a + \frac{b}{T} + cT)$$
 (B-7)

Heat of vaporization relationship,

$$\lambda = \lambda_{O} \left[1 + \frac{T_{S}}{T_{C}} \right]^{\gamma}$$
 (B-8)

where p_{c} is the critical pressure; T_{c} the critical temperature;

The arithmetic mean temperature; p_V the vapor pressure; a,b,c constants; λ , λ heats of vaporization; and T_S the droplet surface temperature. The subscript A refers to the propellant under investigation, the subscript B to the remaining mixture under investigation. Thermodynamic equilibrium is assumed for the mixture B in the film surrounding the droplet at a mean temperature $T = [T_f + T_S]/2$. Dissociation energies are included in the determination of the thermal and transport properties. Average values are determined for the following properties and assumed constant: liquid thermal conductivity, liquid density, liquid specific heat, vapor molecular weight, vapor specific heat, gas mixture apparent molecular weight, gas mixture viscosity, gas mixture thermal conductivity, gas mixture specific heat.

For the case of n-heptane with gaseous oxygen, the thermodynamic properties used are those given in Ref. 18. The thermodynamic properties used in this study are given below:

50/50 Aerozine with NTO

```
1.805 \times 10^{-6} \text{ ft}^2 \text{sec}^{-1}
              56.5 \text{ lbm.ft}^{-3}
              2.44 \times 10^5 lbf ft<sup>-2</sup>
              2.37 \times 10^5 \text{ lbf ft}^{-2}
              1094<sup>0</sup>R
T_{CA}
              629<sup>O</sup>R
T_{CB}
              45.0
M
              22.5
M
              4.30 \times 10^{-5} \text{ lbm ft}^{-1} \text{sec}^{-1}
μ
              0.70306
Pr
              3.54 \times 10^{-5} \text{ Btu ft}^{-1} \text{sec}^{-1} \text{oR}^{-1}
μ<sub>B</sub>
              1.100 Btu lbm<sup>-1</sup>OR<sup>-1</sup>
              4.20 \times 10^{-5} \text{ Btu ft}^{-1} \text{sec}^{-1} \text{O}_{R}^{-1}
              17,167
a
              6381.9
b
```

NTO with 50/50 Aerozine

$$\alpha$$
 1.0378 x 10⁻⁶ ft²sec⁻¹
 ρ 89.34 lbm ft⁻³
 P_{CA} 2.06 x 10⁵ lbf ft⁻²
 P_{CB} 2.37 x 10⁵ lbf ft⁻²
 T_{CA} 776⁰R

 T_{CB} 629⁰R

 M_{A} 46.0 (NO₂ diffuses in film B)

 M_{B} 22.5

 μ 4.30 x 10⁻⁵ lbm ft⁻¹sec⁻¹
 Pr 0.70306

 K_{B} 3.54 x 10⁻⁵ Btu ft⁻¹sec⁻¹o_R⁻¹
 $(c_{p})_{v}$ 2.900 Btu lbm⁻¹o_R⁻¹
 α 2.16 x 10⁻⁵ Btu ft⁻¹sec⁻¹o_R⁻¹

a 17.844

b 6300.0

c 0.0032450

 λ_{O} 268.0

Oxygen with gaseous Hydrogen

$$\alpha$$
 7.5677 x 10⁻⁷ ft²sec⁻¹
 ρ 72.0 lbm ft⁻³
 P_{CA} 1.06 x 10⁵ lbf ft⁻²
 P_{CB} 4.10 x 10⁵ lbf ft⁻²
 P_{CA} 278.16 P_{CA} 1160 P_{CA} 1.00 P_{CA

$$\mu$$
 4.20 x 10⁻⁵ lbm ft⁻¹sec⁻¹

Pr 0.71213

 $K_{\underline{B}}$ 4.04 x 10⁻⁵ Btu ft⁻¹sec⁻¹o_R⁻¹
 $(c_{\underline{p}})_{\underline{V}}$ 2.210 x 10⁻⁵ Btu ft⁻¹sec⁻¹o_R⁻¹

a 16.928

b 1476.50

c 0

Heptane with Gaseous Oxygen

$$\alpha$$
 5.391 x 10⁻⁷ ft²sec⁻¹
 ρ 67.80 lbm ft⁻³
 P_{CA} 5.67 x 10⁴ lbf ft⁻²
 T_{CA} 972^OR

 M_{A} 100.2

 M_{B} 31.0
 μ 5.43 x 10⁻⁵ lbm ft⁻¹sec⁻¹
 Pr 2.0499

 M_{A} 1.29 x 10⁻⁵ Btu ft⁻¹sec⁻¹o_R⁻¹
 M_{B} 2.21 x 10⁻⁵ Btu ft⁻¹sec⁻¹o_R⁻¹

APPENDIX C

CALCULATION PROCEDURE

Method

The procedure used in calculating the droplet evaporation histories and temperature distributions is illustrated by the flow diagram. The droplet surface temperature is given by Eq. (A-11), which requires a knowledge of time averaged thermodynamic amd transport properties which in turn are functions of the droplet surface temperature history. At each succeeding time step the

surface temperature is first assumed constant, thermodynamic and transport properties are calculated and the surface temperature at the end of the time step is determined from Eq. (A-11). The thermodynamic properties are reevaluated with the calculated surface temperature and Eq. (A-11) is solved. The procedure continues until the assumed surface temperature and calculated surface temperature differ by less than 10^{-4} . The surface regression rate, mass evaporation rate, droplet radius, and droplet temperature distribution are then determined. The time is incremented and the procedure continues.

Flow Diagram

Step 1. Load into the machine the following boundary conditions, initial conditions, and computational parameters:

```
Card 1. (I5) KOUNTA - Number of test cases to be analyzed.
```

Card 2. (18A4) TITLE - Title or description of test case.

Card 3. (715) NRUN - Number of test case.

MONTH - Month

MDAY - Day

MYEAR - Year

KA - Maximum number of time steps to be calculated.

JA - Frequency at which the temperature
 distribution is calculated and printed,
 i.e., if JA=10 the temperature distri bution is calculated on every 10th
 step.

NP - Controls frequency of printing droplet
 radius, regression rate, surface
 temperature, etc.

NP=+1 Print the first 100 steps and every JA step thereafter.

NP=-1 Print every step.

Card 4. (8E10.4)

P - Chamber pressure (lbf/ft²)

TF - Flame temperature (OR)

RS - Initial drop radius (ft)

TO - Initial drop temperature (OR)

U - Gas velocity (ft/sec)

DT - Time increment (sec.)

TEND - Droplet lifetime at which calculations
 end (sec)

Card 5. (8E10.4)

CPL - Specific heat-liquid (Btu/lbm R)

RHOL - Liquid density (1bm/ft³)

AKL - Liquid thermal conductivity (Btu/ft secOR)

CPV - Propellant vapor specific heat (Btu/lbmOR)

EMV - Propellant vapor molecular weight

VIS - Combustion gas viscosity (ft-sec/lbm)

CPB - Combustion gas specific heat (Btu/lbmOR)

AKB - Combustion gas thermal conductivity (Btu/ft-secOR)

Card 6. (8E10.4)

PCA - Critical pressure-propellant (.bf/ft²)

PCB - Critical pressure-combustion gases (lbf/ft²)

TCA - Critical temperature-propellant (OR)

TCB - Critical temperature-combustion gases (OR)

EMA - Molecular weight-propellant

EMB - Molecular weight-combustion gases

Card 7. (4E10.4)

PVA, PVB, PVC, PVD - Values of constants A,B,C,D in equation for propellant vapor pressure

$$P_V(T) = \exp (A - \frac{B}{T-C} + DT)$$

Card 8. (4E10.4)

SLA, SLB, SLC, SNl - The first three are values of constants A,B,C in equation for latent heat of vaporization. SNl determines which of two general equation forms is used.

$$SNl = 1$$
 $\lambda (T) = A(1 - \frac{T}{B})^{C}$

$$SN1 = 0$$
 $\lambda (T) = A+BT-CT^2$

Card 9. (4E10.4)

DVA, DVB, DVC, DN1 - The first three are values of constants A,B,C in equation for diffusivity.

DN1 determines which of two general equation forms is used.

DN1=1
$$D_{V} = \frac{2.33(10^{-6})(P_{CA} \cdot P_{CB})^{1/3}}{\sqrt{\frac{M_{A} \cdot M_{B}}{M_{A} + M_{B}}}} \frac{(T_{CA} \cdot T_{CB})^{.416}}{(T_{CA} \cdot T_{CB})^{.416}}$$

$$\left(\frac{\overline{T}}{\sqrt{T_{CA} \cdot T_{B}}}\right)^{1.823}$$

DN1=0
$$D_V = (A+BT+CT^2)(\frac{300}{P})$$

- Step 2. Calculate the initial drop mass, droplet surface regression rate and evaporation rate based upon initial conditions.
- Step 3. Increment time and assume droplet surface temperature equal to that calculated at last time increment.
- Step 4-5-6. Calculate the following time averaged properties: r_s , T_s , T_{film} , P_v . A time averaged property is determined summing the product of the property and time increment over all of the increments and dividing by the total elapsed time. The heat transfer coefficient, mass transfer coefficient and heat of vaporization are then evaluated using the time averaged properties. Equation (A-11) is then solved for surface temperature.
- Step 7,8. The calculated value of surface temperature is compared with the assumed value. If the difference exceeds .0001 the last calculated surface temperature is assumed (Step 8) and steps 4,5,6 are repeated until convergence is obtained.

Step 9. The mass transfer coefficient is recalculated based upon the present values of surface temperature, film temperature, droplet radius and vapor pressure. The instantaneous regression rate

$$\dot{r}_{s} - Kg \frac{P}{\rho_{L}} ln (1 - \frac{P_{V}(T_{s})}{P})$$

and evaporation rate

$$\dot{m} = \rho_{L} 4\pi r_{s}^{2} \dot{r}_{s}$$

are calculated.

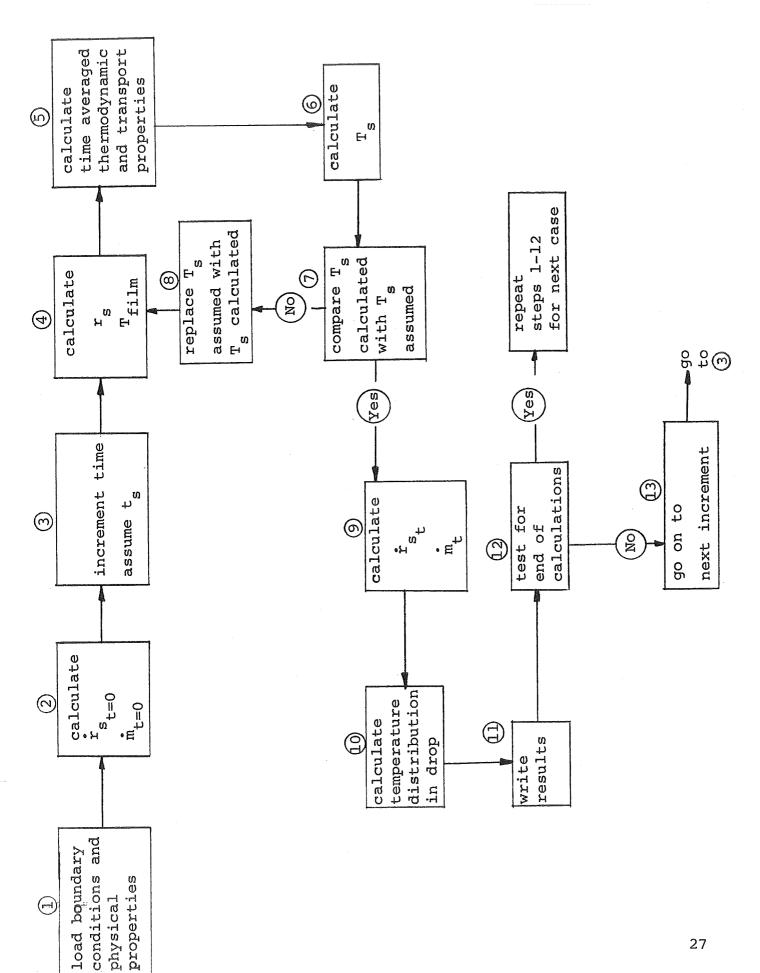
- Step 10. The temperature distribution (T as a function of r/r) in the drop is calculated by solving Eq. (A-ll) at $R=r/r_s=.1,.2,.3.....1.0$ using time averaged properties.
- Step 11. The calculation results for the last time increment are printed. These include: temperature distribution, step number, time, drop radius, radius regression rate, mass evaporation rate, surface temperature, mass transfer coefficient, heat transfer coefficient, Schmidt number, Reynolds number, diffusivity and the ratio of the present drop mass to the initial mass.
- Step 12. Several tests are made to determine if calculations should continue, i.e., should control transfer to step 3 for an additional time increment. If the number of steps already completed is greater than some input number (KA), or if the time is greater than some predetermined quantity (TEND), or if the ratio of present mass to initial mass is less than .9, the calculations on the present test case will terminate and the next set of boundary conditions will be read.

In addition to the above described tests, there are additional checks built into the program. If the solution for surface temperature requires more than 20 iterations, the calculations terminate, a diagnostic message is printed and the next case is read. The calculations will also terminate with an appropriate

message if:

- a) drop vapor pressure is greater than chamber pressure,
- b) drop temperature is greater than critical temperature,
- c) drop radius becomes negative.

The program contains a built-in variation in time increment. The input value of DT (time increment) is used until the time equals 100DT. At that time the time increment is increased by a factor of 10. The time increment is increased whenever the total elapsed time equals 100 times the present time increment.



FLOW DIAGRAM FOR COMPUTER PROGRAM FIG. 13.

```
PROGRAM DROPS (INPUT, OUTPUT, TAPE5=INPUT, TAPE6=OUTPUT)
                PROGRAM DROPS, MAIN
          C
000003
                DIMENSION TSI(2), ERR(2), TITLE(18)
000003
            100 FORMAT(1415)
000003
            101 FORMAT (8E10.4)
            102 FORMAT (1H0, 11H RUN NUMBERIS, 5X, 2HONI3, 1H/I2, 1H/I2//11H STOP AFTERI
-000003
               16,18H STEPS OR AT TIME= E12,4/13H OUTPUT EVERY16,6H STEPS//14H GAS
               2 PRESSURE= E13.5.7H | B/FT2/17H GAS TEMPERATURE= E13.5.8H RANKINE
               3/24H DROPLET INITIAL RADIUS= E13.5,3H FT/21H DROPLET TEMPERATURE=
               4 El3.598H RANKINE/14H GAS VELOCITY= El3.597H FT/SEC/18H DROPLET V
               SELOCITY= E13.5,7H FT/SEC/15H TIME STEPSIZE=
                                                                E13.5,4H SEC)
000003
            103 FORMAT(9H0 ALPHA,7X.4HRHOL,6X.12HGAS CONSTANT,3X,3HPCA,9X.3HPCB,
               19X,3HTCA,9X,3HTCB/ 7E12.4//4X,2HMA,10X,2HMB,10X,2HMV,11X,1HA,11X,
              21HB,9X,3HVIS,9X,2HPR/ 7E12.4//4X.2HKB,8X,5H(CP)V,7X,5HKAPPA/
               2.4)
000003
            104 FORMAT(11H0PV(T)=EXP(F7.3,1H-F8.3,4H/(T-F7.3,2H)+F10.7,3H+T))
000003
            105 FORMAT (16HOITERATION FAILS)
            106 FORMAT(17, 7E13.5, 3E9.2, F6.3)
-000003
000003
            107 FORMAT(1H0,5X,1HK,5X,4HTIME,10X,2HRS,10X,5HRSDOT,9X,4HWDOT,10X,2HT
               15,10x,2HKG,11x,1HH,8X,3HSCH,8X,3HREY,8X,2HDV,3X,4HM/MI/1H0)
-000003
            108 FORMAT(F35.1, E15.5)
            109 FORMAT (1H0///17x,47HTEMPERATURE DISTRIBUTION IN THE DROPLET AT STE
000003
               1PI6//30X,4HR/RS,12X,1HT)
-000003
            110 FORMAT(1H1,30X,25HPROPULSION SCIENCES, INC./36X,12HP.O. BOX 814/
               133X,18HMELVILLE, NEW YORK//22X,45HWORK PERFORMED UNDER NASA CONTRA
               2CT NAS3-12030
            111 FORMAT(1H0///17X,52HDROP VAPOR PRESSURE IS OREATER THAN CHAMBER PR
-000003
               1ESSURE
000003
            112 FORMAT (18A4)
-000003
            113 FORMAT(1H0//20X,18A4)
000003
            114 FORMAT(11HnLAMBDA(T)=F7.2,1H+F8.4,3H#T-F10.7,5H#T##2)
            115 FORMAT(11HOLAMBDA(T)=F7.2.6H(1.-T/F7.2.3H) **F7.4)
000003
<del>-000003</del>
            <del>116 FORMAT(83HoD(PyT)=(PCA*PCB)**。33/SQRT(MA*MB/(MA*MB))*(TCA*TCB)**5/</del>
               112/P#A#(T/SQRT(TCA#TCB))##B)
000003
            117 FORMAT(9H0D(P,T)=(E13.5,1H+E13.5,3H*T+E13.5,13H*T**2)*30\sqrt{P}
-000003
            118 FORMAT(34HnDROP RADIUS IS NEGATIVE AT TIME =E13.5)
            119 FORMAT (55HODROP TEMPERATURE CAUSING VAPOR PRESSURE FCN TO BLOW UP)
000003
E00000
            120 FORMAT (47HOMASS REMAINING TS LESS THAN .9 OF INITIAL MASS)
000003
                PV(X) = EXP(PVA - PVB/(X - PVC) + PVD + X)
000017
                D1(x,Y) = (PCA+PCB) ** ** * *333333/SQRT(EMA+EMB/(EMA+EMB)) ** (TCA+TCB) ** ** * *416
                <del>1667/x*AA*(Y/SQRT(TCA*TCB))**BB</del>-
000056
                D2(X,Y)=(DVA+DVB*Y+DVC*Y**2.)*300./X
 000072
                SL1(X) = SLA + (1.-X/SLB) + SLC
000103
                SL2(X)=SLA+SLB*X-SLC*X**2.
000115
                D(X,Y)=DN_1*D_1(X,Y)*ABS(DN_1-1*)*D_2(X,Y)
000135
                RR=1545.
000136
                AA=2.33E-6
000140
                88=1.823
          C
-000141
                KOUNT=0
 000143
                READ (5,100) KOUNTA
000150
              1 KOUNT=KOUNT+1
                IF (KOUNT. GT. KOUNTA) CALL EXIT
-000152
000155
                READ (5,112) TITLE
```

```
000163
                  READ (5, 100) NRUN, MONTH, MDAY, MYEAR, KA, JA, NP
                  READ (5,101) P, TF, RS, TO, U, V, DT, TEND, CPL, RHOL, AK, CPV, EMV, VIS, CPM, AKB
 000205
                 1,PCA,PCB,TCA,TCB,EMA,EMB
 000265
                  READ (5,101) PVA, PVB, PVC, PVD
                  READ (5,101) SLA, SLB, SLC, SN1
 000301
 000315
                  READ (5,101) DVA, DVB, DVC, DN1
                  ALPHA=AK/(RHOL*CPL)
 000331
                  PR=CPM*VIS/AKB
 000333
000336
                  WRITE (6,110)
                  WRITF(6,113)(TITLE(I),I=1,18)
 000342
                  WRITE (6,102) NRUN, MONTH, MDAY, MYEAR, KA, TEND, JA, P, TF, RS, TO, U, V, DT
 000354
                  WRITF(6,103)ALPHA,RHOL,RR,PCA,PCB,TCA,TCB,EMA,EMB,EMV,AA,BB,VIS.PR
000414
                 1 , AKB , CPV , AK
 000462
                  WRITE (6,104) PVA, PVB, PVC, PVD
                  IF (SN1.LT..5) WRITE (6,114) SLA, SLB, SLC
000476
                  IF (SN1.GT..5) WRITE (6,115) SLA, SLB, SLC
 000512
 000526
                  IF (DN1.GT..5) WRITE (6,116)
                  IF (DN1-LT-.5) WRITE (6,117) DVA, DVB.DVC
<del>-000534</del>-
 000550
                  TCRTT=100.*DT
                  SSAMJ=4.*3.14159*RS**3.*RHOL/3.
 000552
 000560
                  DUM8=12.5663704*RHOL
                  DUM2=.6*PR **.333333
 000562
<del>-000566</del>
                  DUM3=2. *ABS(U-V)
 000572
                  DUM4=EMV/2 /RR
 000575
                  K=0
                  J=0
 000576
 000577
                  TIMF=0.
 000577
                  TS=Tn
                  TSSUM=TS
 000601
                  TAVSUM=(TF+T0) +.5
 000602
 000604
                  RSSUM=RS
 000606
                  ROBAR=P#EMB/(RR#TAVSUM)
 000611
                  VORSUM=VIS/ROBAR
 000613
                  SCH=VORSUM/D(P,TAVSUM)
                  DUM1 = . 6 * SCH * * . 333333
<del>-000617</del>
 000623
                  PVSUM=PV(To)
                  DUM=1 .- PVSUM/P
 000626
000631
                  IF (DHM.LE.1.E-10) GO TO 11
                  AKG=(2.+DUM1*SQRT(RS*DUM3/VORSUM))*D(P,T0)*DUM4/RS/TAVSUM
 000634
 000652
                  RSDOT= AKG*P*ALOG(1.-PVSUM/P)/RHOL
                  WDOT=DUM8*RS**2*RSDOT
<del>-000663</del>
 000667
                  WRITF (6,107)
                  WRITE (6,106)K, TIME, RS, RSDOT, WDOT, TS
 000672
                  RSINT=0.
<del>-000712</del>
 000713
                  VORINT=0.
 000713
                  TAVINT=0.
-000714
                  PVINT=0.
                  TSINT=0.
 000715
-000717
                2 K=K+1
 000721
                  J=J+1
                  TIME=TIME+DT
 000722
000724
                  KIP=1
 000724
                  ME=1
 000725
                  TS1(1)=TS
000727
                  TS1(2)=0.9*TS
                3 TAV=(TF+TS1(ME))+.5
 000731
                                                                                  29
```

```
000734
                DD=D(P,TAV)
000740
                ROAV=P*EMB/(RR*TAV)
000744
                VOR=VIS/ROAV
000745
                RS1=RS+RSDOT*DT
000751
                IF (RS1.LE.n.0) GO TO 12
                RSBAR=(RSINT+RS1*DT)/TIME
000753
000755
                VORBAR=(VORINT+VOR*DT)/TIME
                IF (ABS(PVA-PVB/(TS1(ME)-PVC)+PVD*TS1(ME)), GT.300.) GO TO 13
000760
000773
                PVBAR=(PVINT+PV(TS1(ME))*DT)/TIME
001001
                TAVBAR=(TAVINT+TS1(ME)+DT)/TIME
001004
                TSBAR=(TSINT+TS1(ME)+DT)/TIME
                H=AKB/2./RSBAR*(2.*DUM2*SQRT(DUM3*RSBAR/VORBAR))
001010
001021
                DUM1 = .6* (VORBAR/D (P; TAVBAR)) ** .333333
                AKGRAR=(2.+DUM1*SQRT(RSBAR*DUM3/VORBAR))*D(P.TAVBAR)*DUM4/RSBAR/TA
001031
               IVBAR-
001050
                DUM=1 .- PVBAR/P
001054
                IF (DUM.LE.1.E-10) GO TO 11
001057
                DUM= (AKGBAR*P*ALOG(DIJM))/AK
                C=H/AK+CPV*DUM
001063
001067
                IF (SN1.LT..5) HEVAP=SL2 (TSBAR)
                IF (SN1.GT.,5) HEVAP=SL1 (TSBAR)
001074
001101
                B=-HEVAP*DUM-C*TF
                A=C-1./RSBAR
001105
                DUMS=SQRT(ALPHA*TIME)/RSBAR
001107
                DUM6=DUM5*A*RSBAR
001114
                DUM7=DUM6*#2
001116
001117
                DUM9=EXP(DUM7) -
001121
                TSA=T0+(T0+(B+T0/RSBAR)/A)*(ERFC(1./DUM5)-ERFC(0.)+DUM9*(ERFC(DUM6
               1) - Exp(2.*A*RSBAR) *ERFC(1./DUM5+DUM6)))
001161
                ERR (ME) = TSA-TS1 (ME)
                IF (ARS(ERR(ME)).LE.1.E-4)GO TO 4
001164
001170
                IF (ME.EQ.2) GO TO 5
001172
                ME=2
001173
                GO TO 3
001173
              5 TSAA=TS1(1)-ERR(1)*(TS1(2)-TS1(1))/(ERR(2)-ERR(1))
001202
                ERR(1) = ERR(2)
001203
                TSl(1) = TSl(2)
001204
                TS1(2)=TSAA
001206
                KIP=KIP+1
                IF (KIP.LE.20) GO TO 3
001207
001212
                WRITE (6,105)
001215
                60 Th 1
         C
001216
             4 RS=RS1
001217
                RSINT=RSINT+RS*DT
001222
                TS=TS1 (ME)
001224
                DUM1 = .6 + (VOR/DD) + + . 333333
                AKG= (2.+DUM1*SQRT(RS*DUM3/VOR))*DD*DUM4/RS/TAV
001231
                IF (ABS(PVA-PVB/(TS-PVC)+PVD*TS).GT.300.) GO TO 13
001244
001256
                DUM=1.-PV(TS)/P
                IF (DUM. LE. 1.E-10) 60 TO 11
001262
001265
                RSDOT=AKG*P*ALOG(DUM)/RHOL
001271
                TAVINT=TAVINT+TAV*DT
                VORINT=VORINT+VOR#DT
001275
001277
                TSINT=TSINT+TS*DT
001302
                PVINT=PVINT+PV(TS) *DT
                WDOT=DUM8*RS**2*RSDOT
001306
001310
                SCH=VOR/DD
```

```
001312
                 REY=2. *RS*ABS(U-V)/VOR
001320
                 SSAM=4.*3.14159*RS**3.*RHOL/3.
001325
                 RAT=SSAM/SSAMI
                 IF (TIME.GT. TCRIT. AND. NP.GT. 0) GO TO 10
001327
                 WRITE (6,106) K, TIME, RS, RSDOT, WDOT, TS, AKG, H, SCH, REY, DD, RAT
001341
001374
              10 IF (RS.LE.O.) TEND=TIME
                 IF (J.LT. JA. AND. TIME. LT. TEND) GO TO 6
001400
          C
               8 J=0
001412
001413
                 TT=TIME/DT+.5
001416
                 IF (TT.GT.100.) DT=10.*DT
001422
                 WRITF (6, 109)K
                 DO 7 N=1,10
001430
                 R=.] *FLOAT(N)
001432
001435
                 DOM1 = (1 .+ R) /2 . / DUM5
001440
                 DOM2=(1.-R)/2./DUM5
                 T=T0+(T0+(R+T0/RSBAR)/A)/R*(ERFC(DOM1)-ERFC(DOM2)+DUM9*(EXP(A*RSBA
001442
                1R#(1.-R))#ERFC(DOM2+DUM6)-EXP(A*RSBAR*(1.+R))#ERFC(DOM1+DUM6))}
               7 WRITE (6,108) R,T
001512
                 WRITE (6,107)
001524
001530
                 IF (NP.LT.O) GO TO 6
001532
                 WRITF (6,106) K, TIME, RS, RSDOT, WDOT, TS, AKG, H, SCH, REY, DD, RAT
001565
                 IF (RAT.LT.,9) GO TO 14
               6 IF (K.GT.KA.OR.TIME.GE.TEND) GO TO 1
001570
001603
                 GO TO 2
001603
              11 WRITE (6,111)
001607
                 <del>60 To 1</del>
001610
              12 WRITE (6,118) TIME
001616
                 GO TO 1
<del>001617</del>
              13 WRITF (6,119)
001623
                 GO TO 1
001624
              14 WRITE (6,120)
001630
                 60 TO 1
          C
               . END
001631
```

APPENDIX D

SYMBOLS

```
Α
        factor, Eq. (12)
A_d
        droplet surface area
        film thickness factor, Eq. (12), Eq. (A-7a)
        constants, Eq. (6)
a,b,c
C
        constant, Eq. (13), (A-9a)
        average droplet vapor specific heat
        molecular diffusion coefficient, factor, Eq. (A-7b),
        diameter
h
        film heat transfer coefficient, enthalpy
K
        evaporative constant, Eq. (20)
Κq
        mass transfer coefficient
Km
        thermal conductivity of vapor-gas mixture
M
        molecular weight
Pr
        Prandtl number
        pressure; transform variable
р
        defined in Eq. (A-5c), heat transfer
q
R
        ratio r/r
        Reynolds number
Re
R_{O}
        universal gas constant
r
        droplet radius
r<sub>s</sub>
        regression rate of droplet surface
S
        entropy
Sc
        Schmidt number
\mathbf{T}
        temperature
\overline{\mathbf{T}}
        average temperature [T_f+T_g]/2
t
U
        function defined in Eq. (A-1)
u
        gas velocity
        droplet velocity
v
ŵ
       mass evaporation rate
X
       mole fraction
       mass fraction
У
        thermal diffusivity
α.
        exponent, Eq. (7)
Υ
       gas density
ρ
       droplet thermal conductivity
\varkappa
λ
       heat of vaporization
```

```
λ_{O} constant, Eq. (7)
μ viscosity
ρ density
αt/r_{S}^{2}
θ transport property
φ Eq. (B-5)
```

Subscripts

```
mixture
m
sh
       superheat
       evaporation
λ
Ł
       liquid
       total
\mathbf{T}
       surface
s
       initial, reference
0
f
       flame
       vapor
v
       critical, chamber
С
       radius, differentiation with respect to radius
r
t
       time, differentiation with respect to time
       specie index
i,j
```

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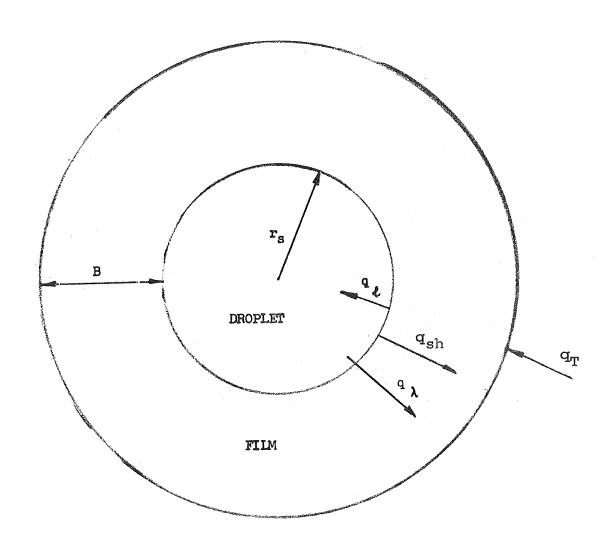


FIG. 1. SCHEMATIC DIAGRAM OF HEAT TRANSFER TO VAPOR FILM AND LIQUID DROPLET

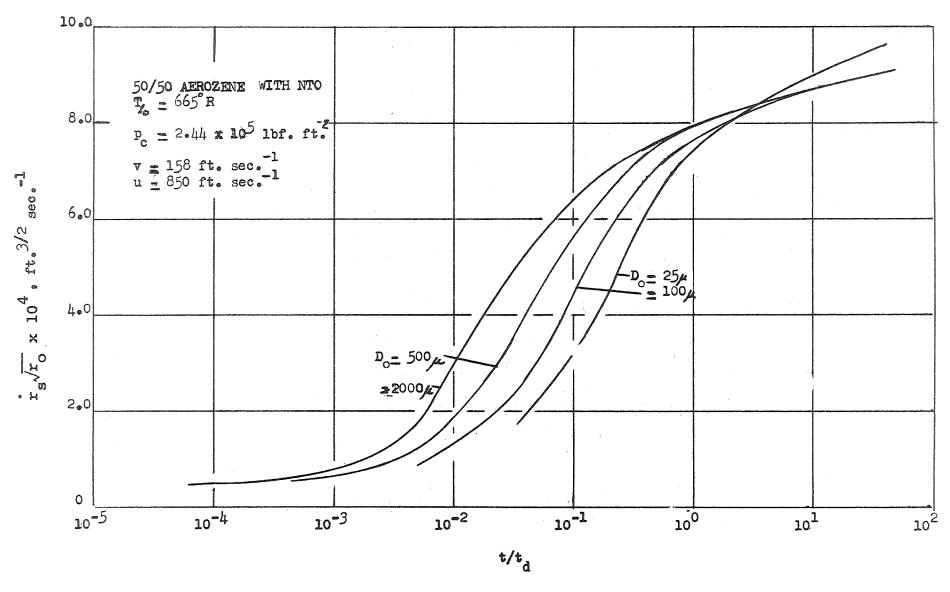


FIG. 2a. SURFACE REGRESSION RATE VS. TIME FOR VARIATION IN INITIAL DROPLET RADIUS, 50/50 AEROZINE WITH NTO

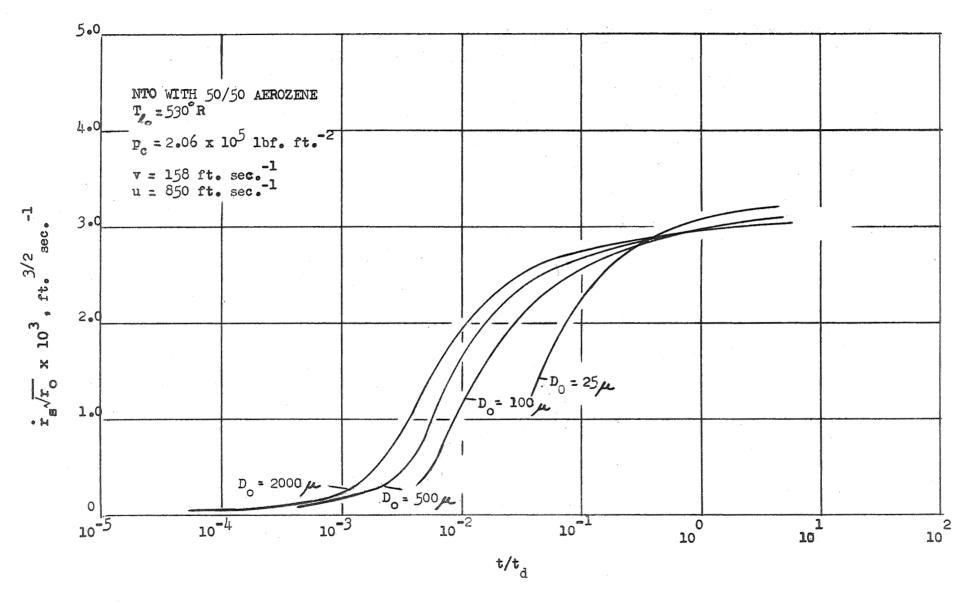


FIG. 2b. SURFACE REGRESSION RATE VS. TIME FOR VARIATION IN INITIAL DROPLET RADIUS, NTO WITH 50/50 AEROZINE

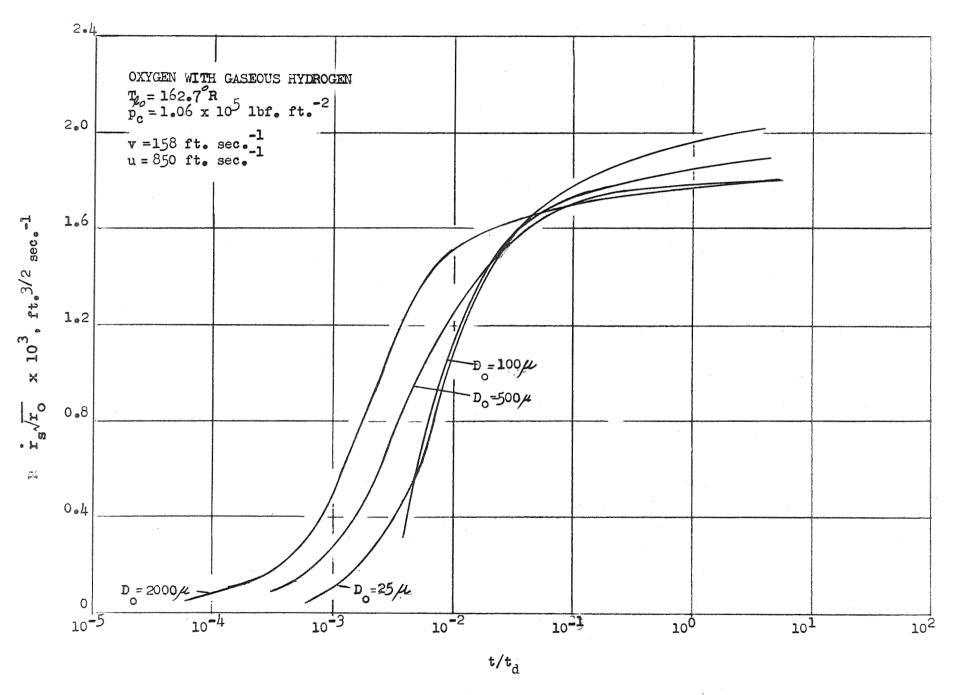


FIG. 2c. SURFACE REGRESSION RATE VS. TIME FOR VARIATION IN INITIAL DROPLET RADIUS, OXYGEN WITH GASEOUS HYDROGEN

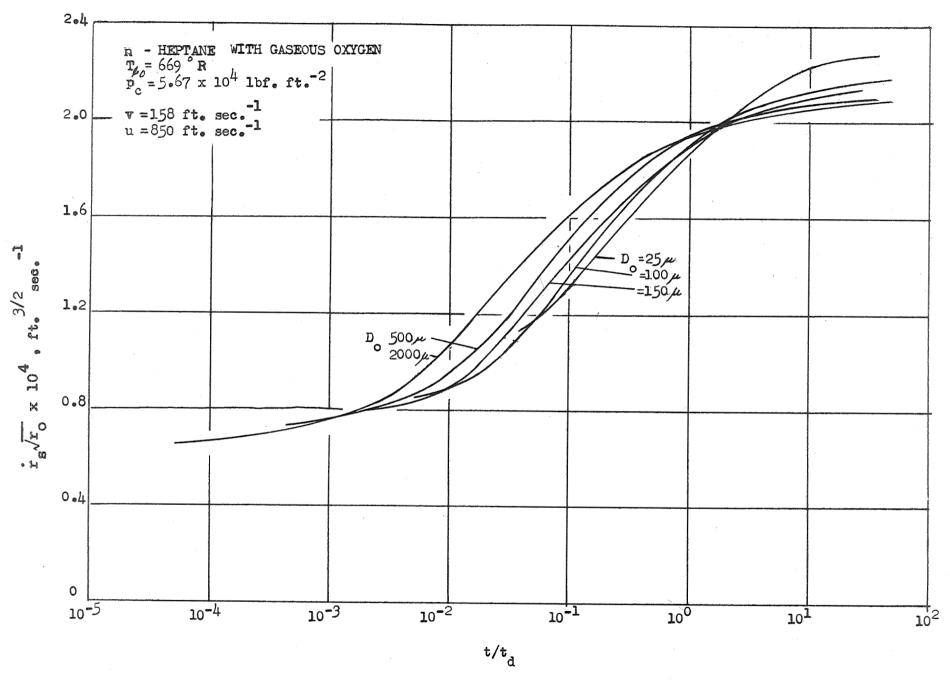


FIG. 2d. SURFACE REGRESSION RATE VS. TIME FOR VARIATION IN INITIAL DROPLET RADIUS, n-HEPTANE WITH GASEOUS OXYGEN

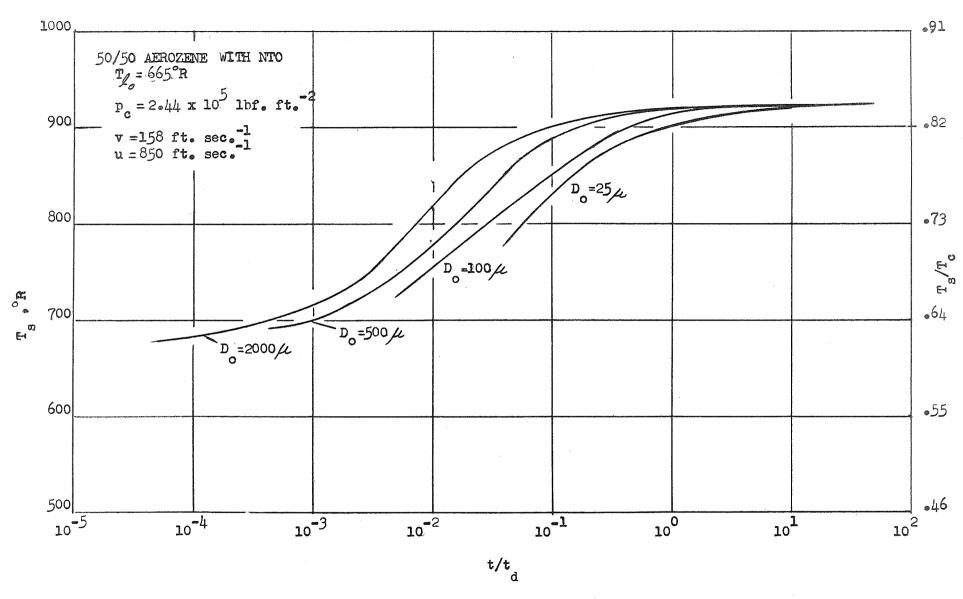


FIG. 3a. SURFACE TEMPERATURE VS. TIME FOR VARIATION IN INITIAL DROPLET RADIUS, 50/50 AEROZINE WITH NTO

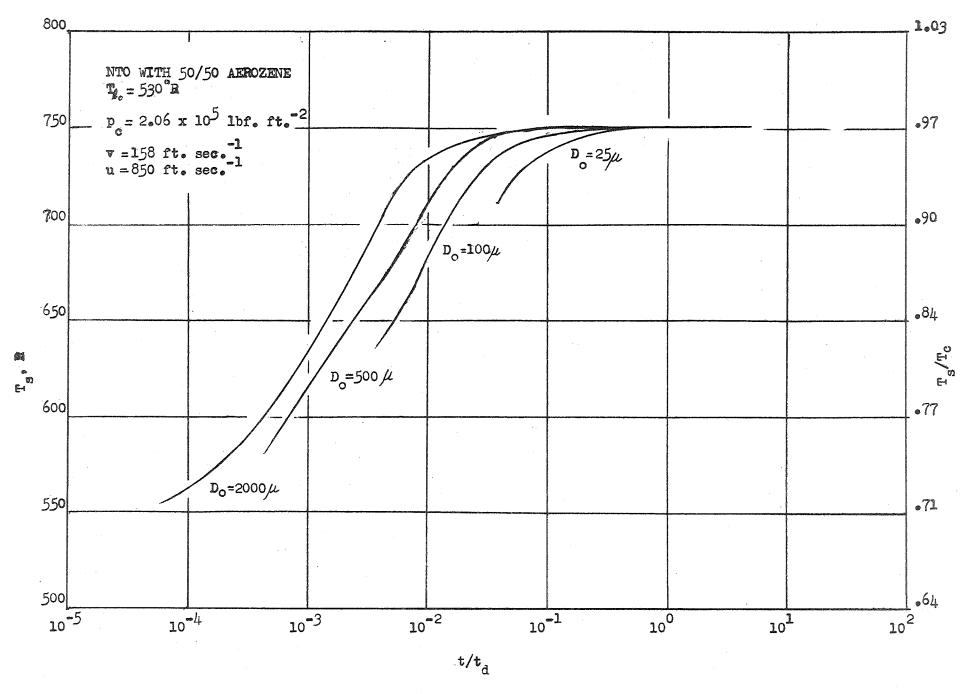


FIG. 3b. SURFACE TEMPERATURE VS. TIME FOR VARIATION IN INITIAL DROPLET RADIUS, NTO WITH 50/50 AEROZINE

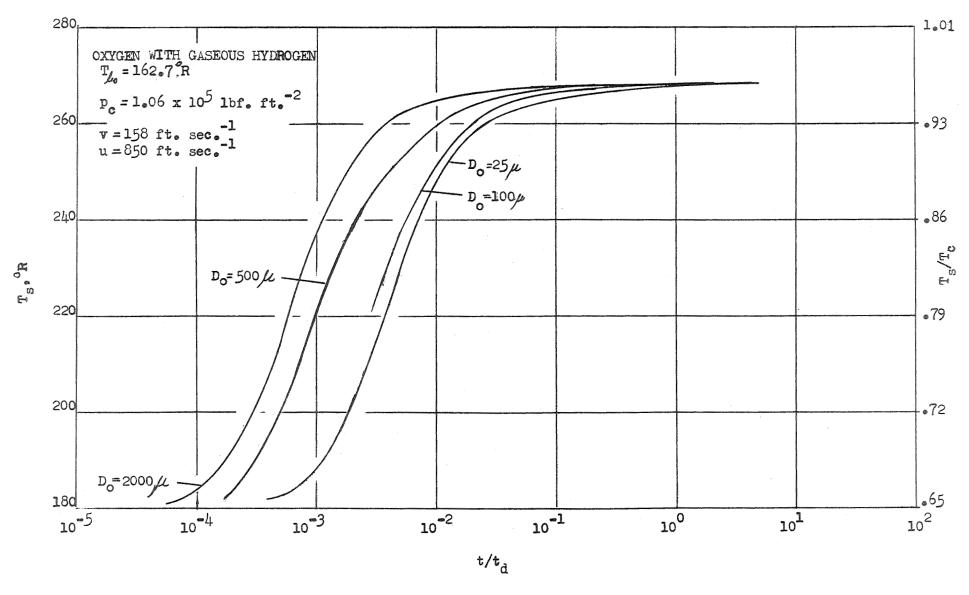


FIG. 3c. SURFACE TEMPERATURE VS. TIME FOR VARIATION IN INITIAL DROPLET RADIUS, OXYGEN WITH GASEOUS HYDROGEN

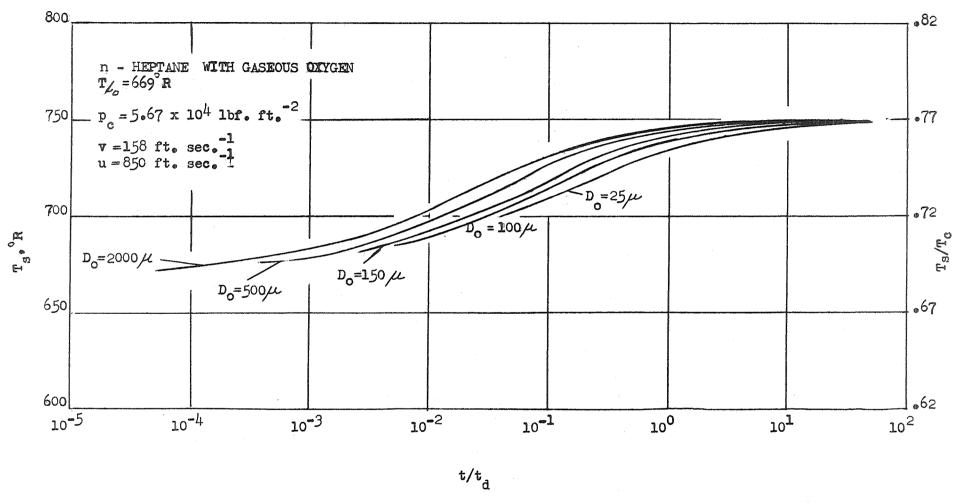


FIG. 3d. SURFACE TEMPERATURE VS. TIME FOR VARIATION IN INITIAL DROPLET RADIUS, n-HEPTANE WITH GASEOUS OXYGEN

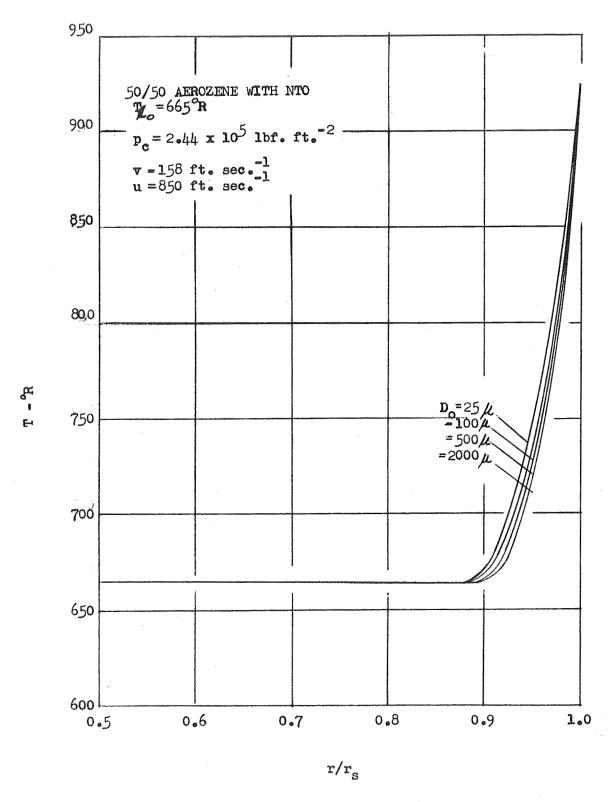


FIG. 4a. DROPLET TEMPERATURE DISTRIBUTION FOR VARIATION IN INITIAL DROPLET RADIUS, AT A TIME 10⁻⁴ SECOND, 50/50 AEROZINE WITH NTO

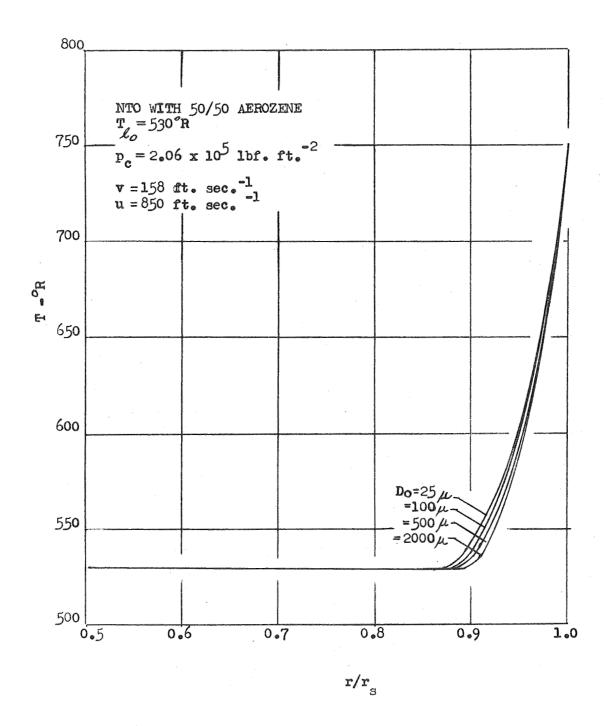


FIG. 4b. DROPLET TEMPERATURE DISTRIBUTION FOR VARIATION IN INITIAL DROPLET RADIUS, AT A TIME CV 10^{-4} SECOND, NTO WITH 50/50 AEROZINE

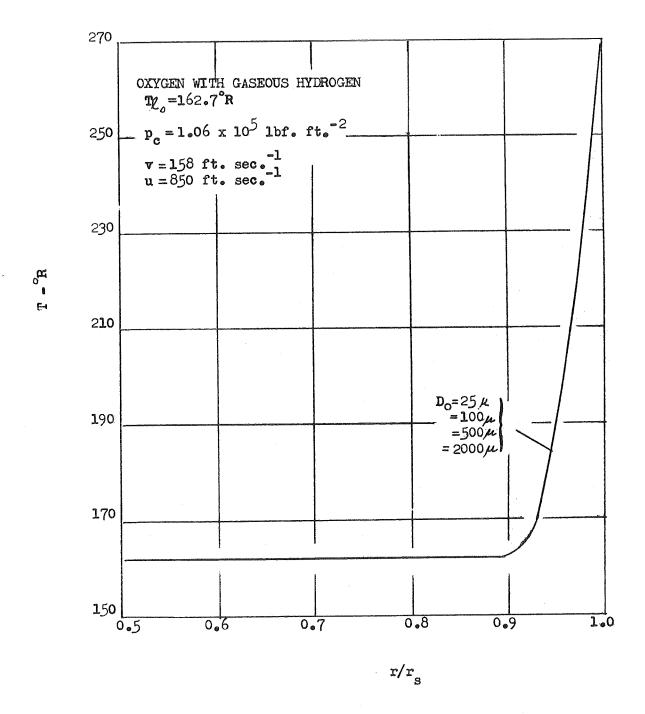


FIG. 4c. DROPLET TEMPERATURE DISTRIBUTION FOR VARIATION IN INITIAL DROPLET RADIUS, AT A TIME OF 10-4 SECOND, OXYGEN WITH GASEOUS HYDROGEN

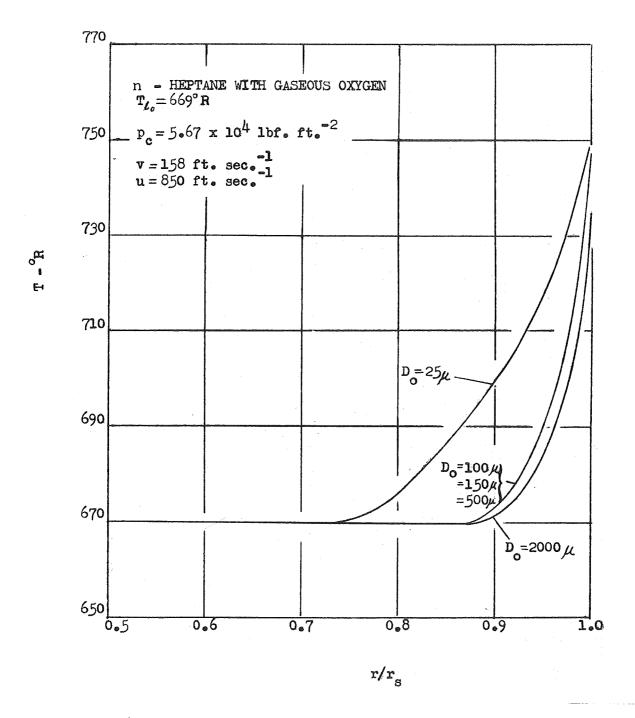


FIG. 4d/ DROPLET TEMPERATURE DISTRIBUTION FOR VARIATION IN INITIAL DROPLET RADIUS, AT A TIME OF 10^{-4} SECOND, n-HEPTANE WITH GASEOUS OXYGEN

FIG. 5a. SURFACE REGRESSION RATE VS. TIME FOR VARIATION IN INITIAL DROPLET TEMPERATURE, 50/50 AEROZINE WITH NTO

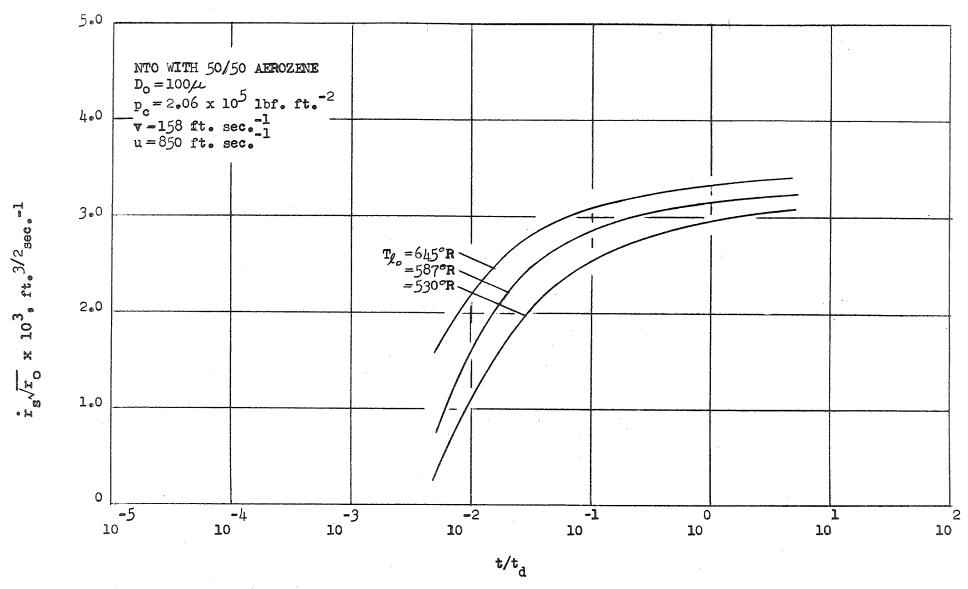


FIG. 5b. SURFACE REGRESSION RATE VS. TIME FOR VARIATION IN INITIAL DROPLET TEMPERATURE, NTO WITH 50/50 AEROZINE

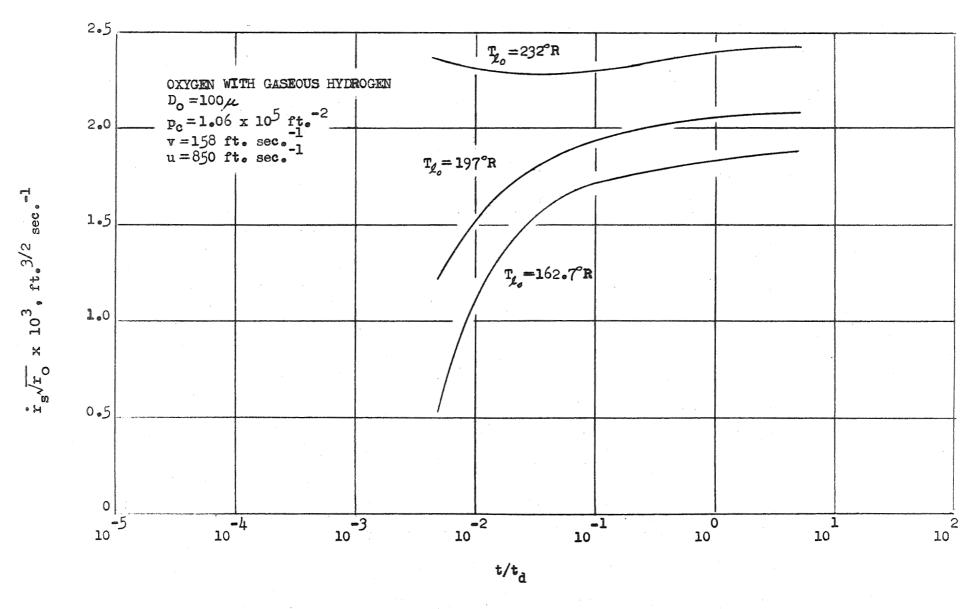


FIG. 5c. SURFACE REGRESSION RATE VS. TIME FOR VARIATION IN INITIAL DROPLET TEMPERATURE, OXYGEN WITH GASEOUS HYDROGEN

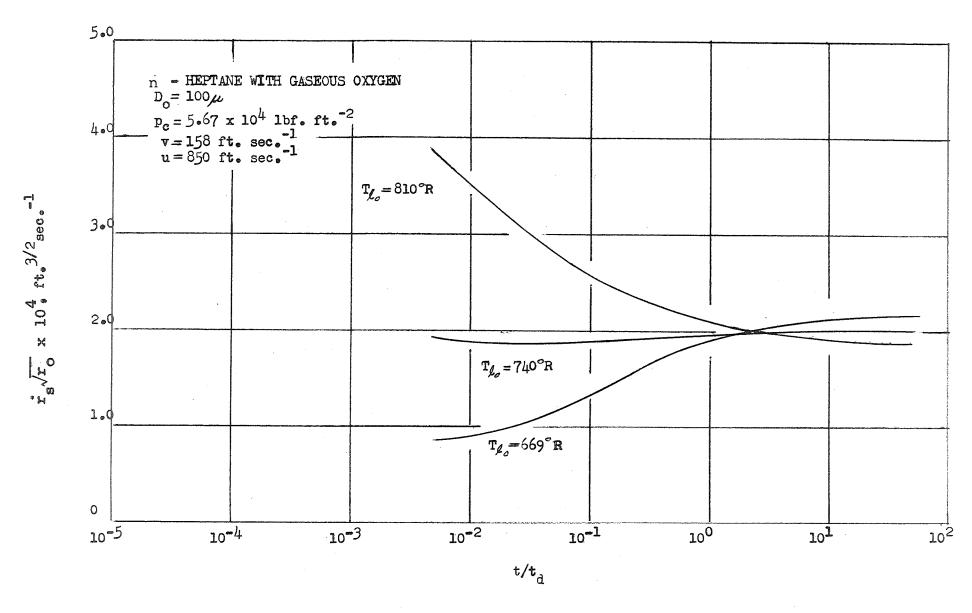


FIG. 5 d. SURFACE REGRESSION RATE VS. TIME FOR VARIATION IN INITIAL DROPLET TEMPERATURE, n-HEPTANE WITH GASEOUS OXYGEN

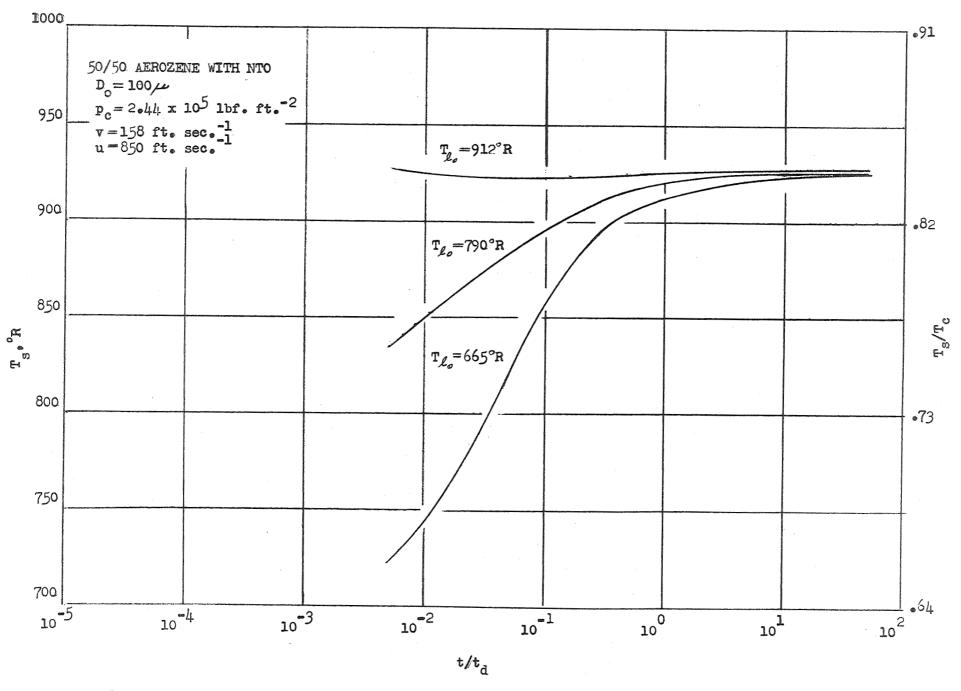


FIG. 6a. SURFACE TEMPERATURE VS. TIME FOR VARIATION IN INITIAL DROPLET TEMPERATURE, 50/50 AEROZINE WITH NTO

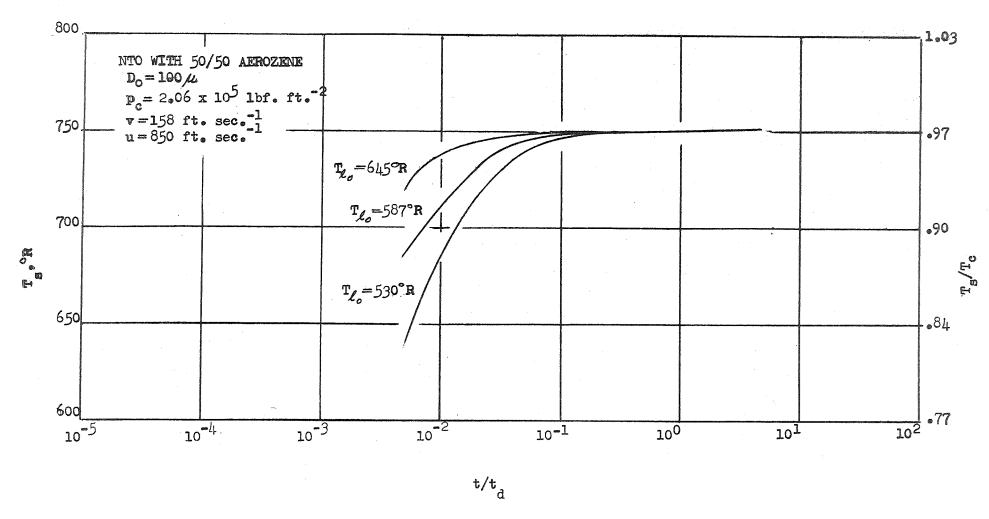


FIG. 6b. SURFACE TEMPERATURE VS. TIME FOR VARIATION IN INITIAL DROPLET TEMPERATURE, NTO WITH 50/50 AEROZINE

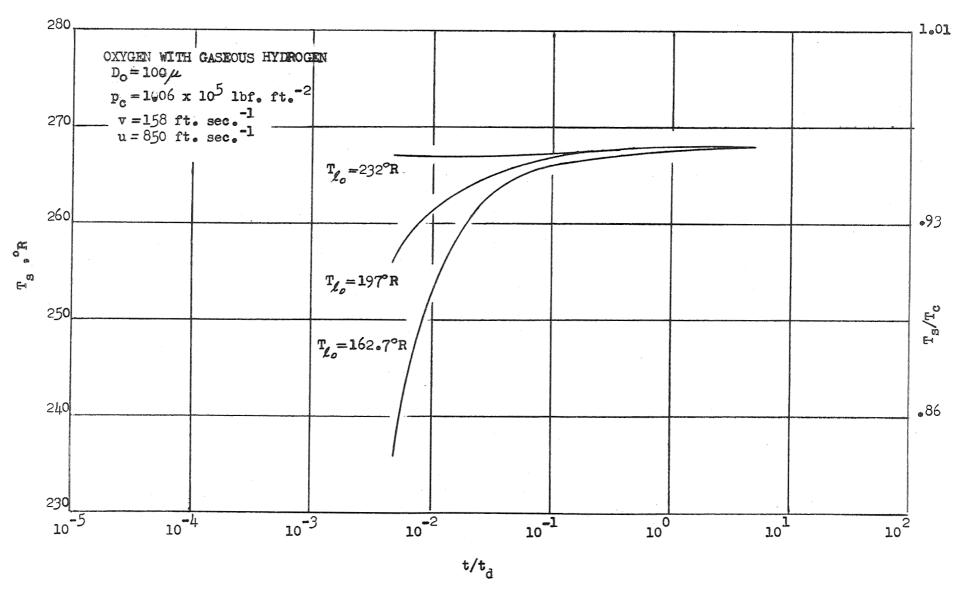


FIG. 6c. SURFACE TEMPERATURE VS. TIME FOR VARIATION IN INITIAL DROPLET TEMPERATURE, OXYGEN WITH GASEOUS HYDROGEN

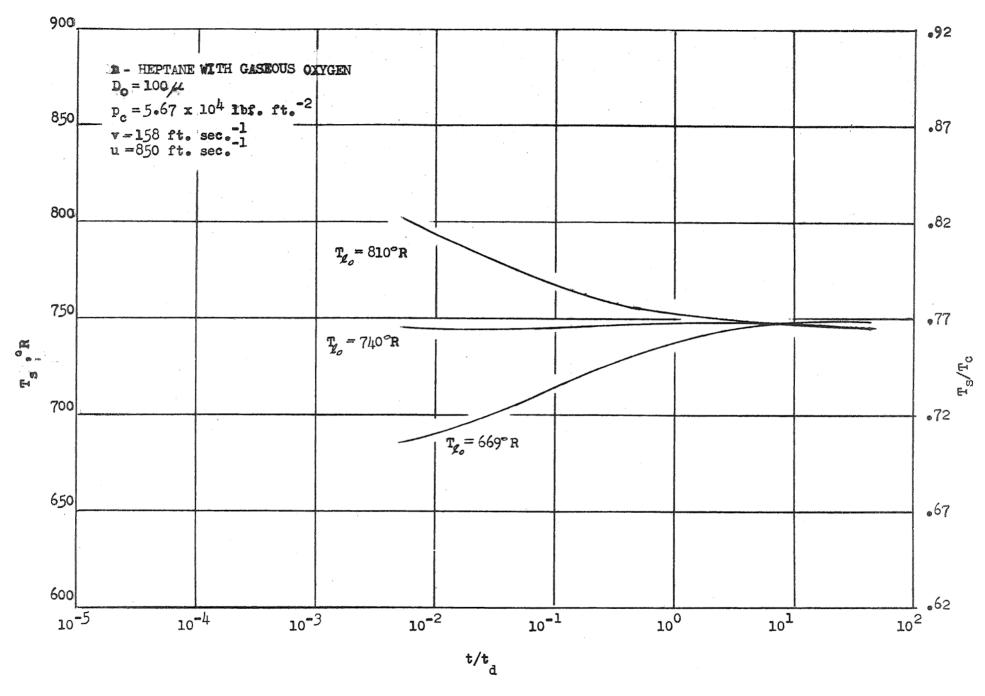


FIG. 6d. SURFACE TEMPERATURE VS. TIME FOR VARIATION IN INITIAL DROPLET TEMPERATURE, n-HEPTANE WITH GASEOUS OXYGEN

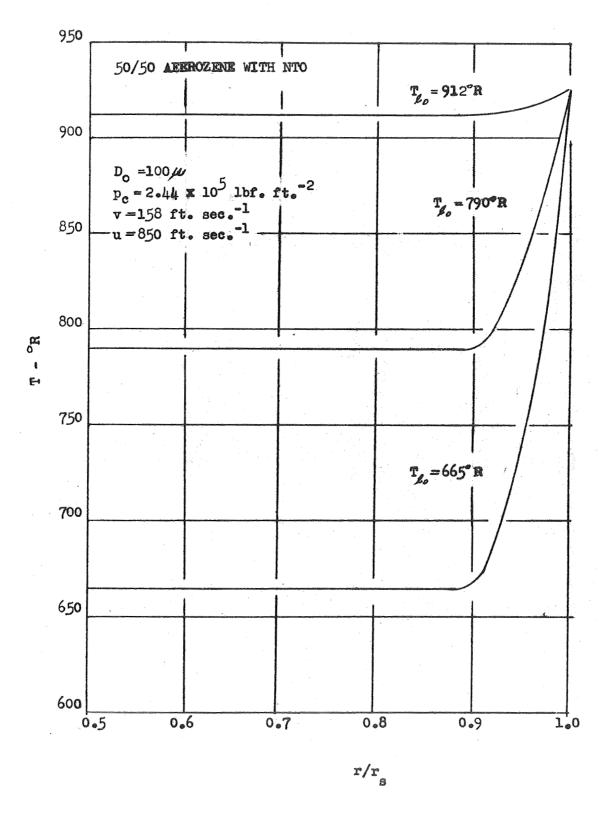


FIG. 7a. DROPLET TEMPERATURE DISTRIBUTION FOR VARIATION IN INITIAL DROPLET TEMPERATURE, AT A TIME OF 10^{-4} SECOND, 50/50 AEROZINE WITH NTO

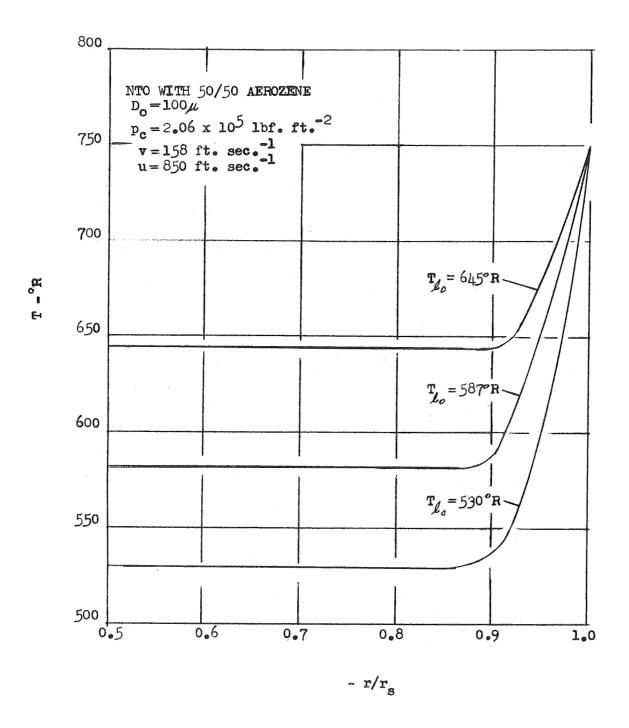


FIG. 7b. DROPLET TEMPERATURE DISTRIBUTION FOR VARIATION IN GNITIAL DROPLET TEMPERATURE, AT A TIME OF 10^{-4} SECOND, NTO WITH 50/50 AEROZINE

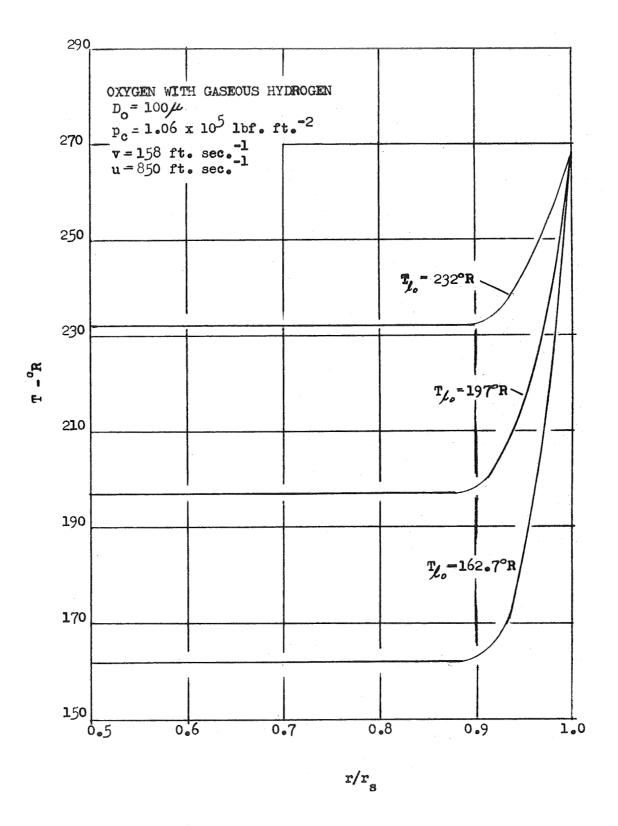


FIG. 7c. DROPLET TEMPERATURE DISTRIBUTION FOR VARIATION IN INITIAL DROPLET TEMPERATURE, AT A TIME OF 10-4 SECOND, OXYGEN WITH GASEOUS HYDROGEN

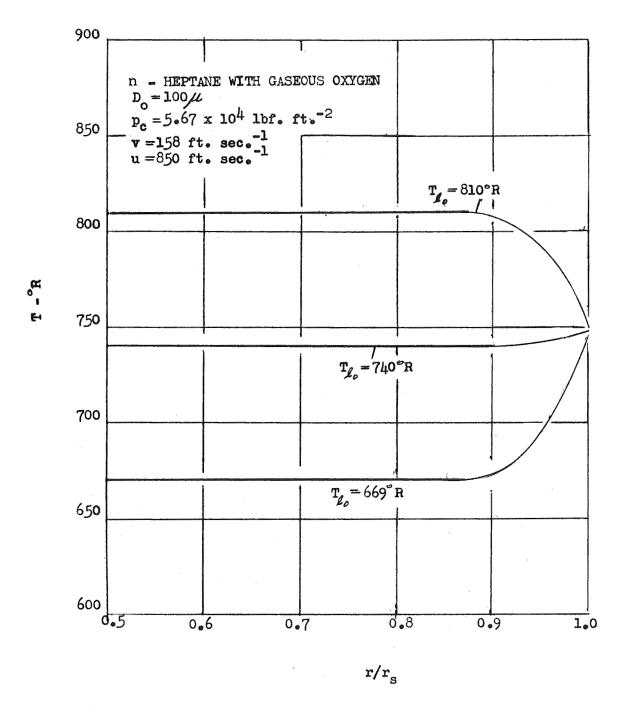


FIG. 7d. DROPLET TEMPERATURE DISTRIBUTION FOR VARIATION IN INITIAL DROPLET TEMPERATURE, AT A TIME OF 10^{-4} SECOND, n-HEPTANE WITH GASEOUS OXYGEN

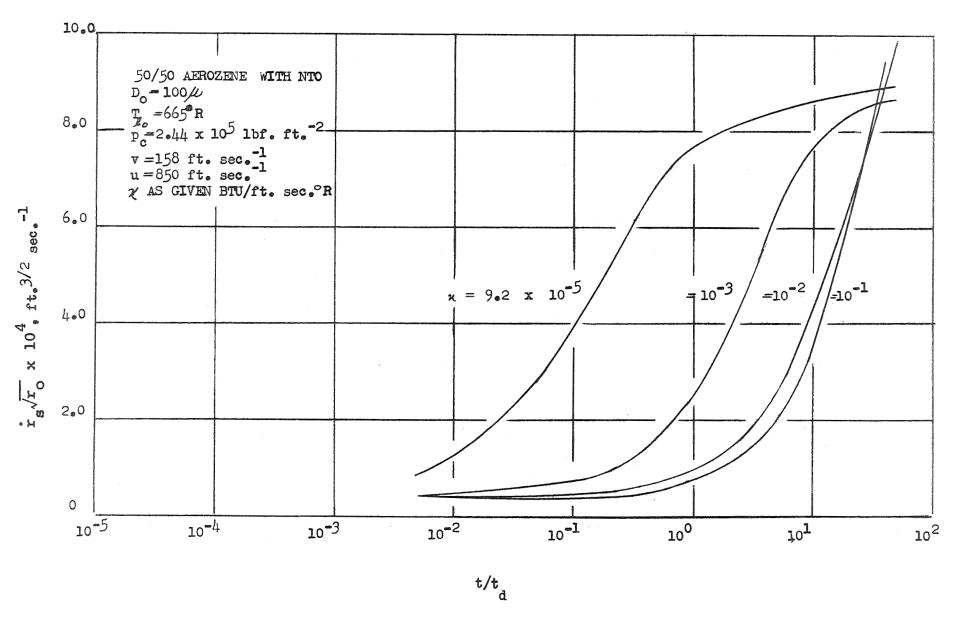


FIG. 8a. SURFACE REGRESSION RATE VS. TIME FOR VARIATION IN PROPELLANT THERMAL CONDUCTIVITY, 50/50 AEROZINE WITH NTO

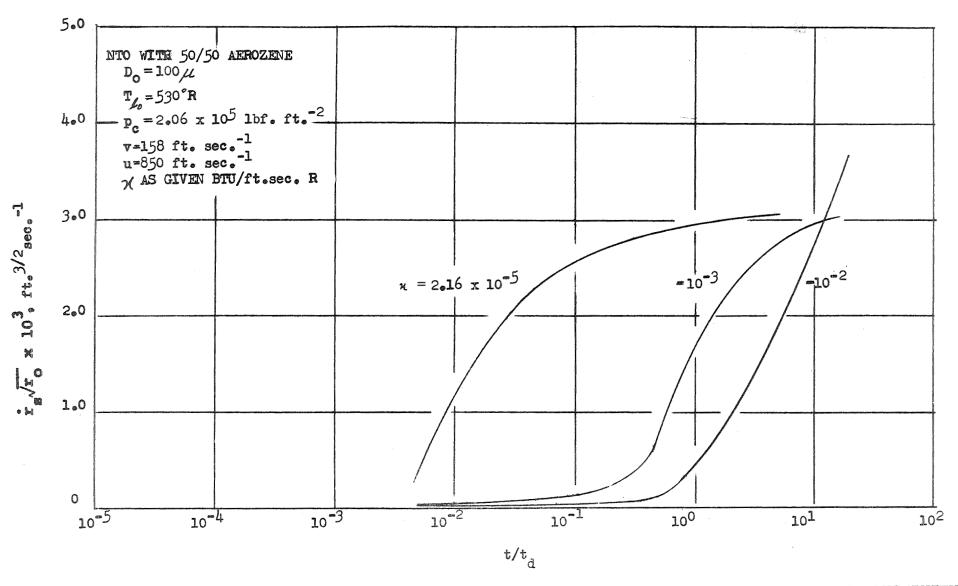


FIG. 8b. SURFACE REGRESSION RATE VS. TIME FOR VARIATION IN PROPELLANT THERMAL CONDUCTIVITY, NTO WITH 50/50 AEROZINE

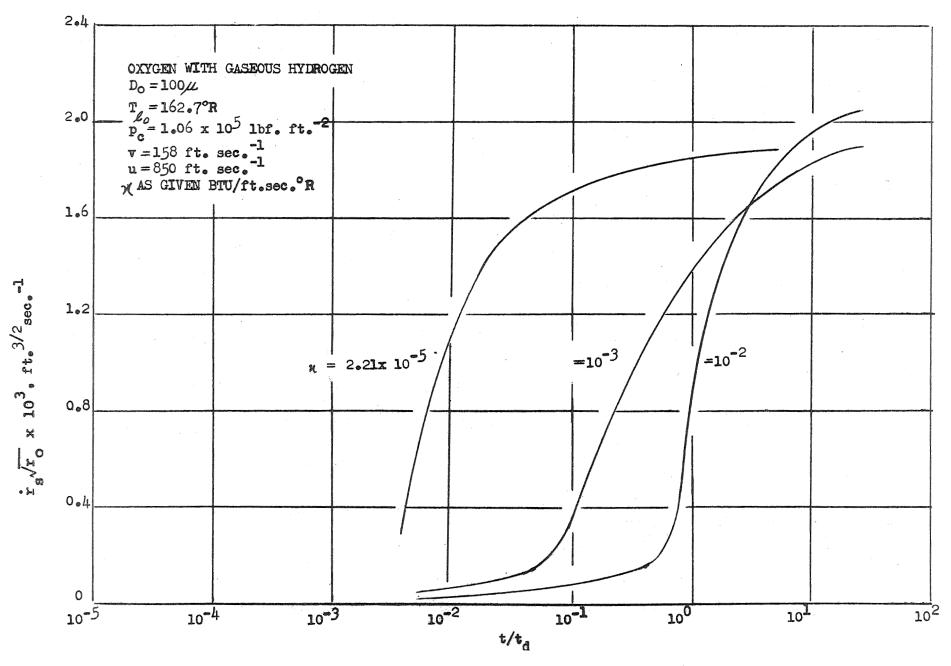


FIG. 8c. SURFACE REGRESSION RATE VS. TIME FOR VARIATION IN PROPELLANT THERMAL CONDUCTIVITY, OXYGEN WITH GASEOUS HYDROGEN

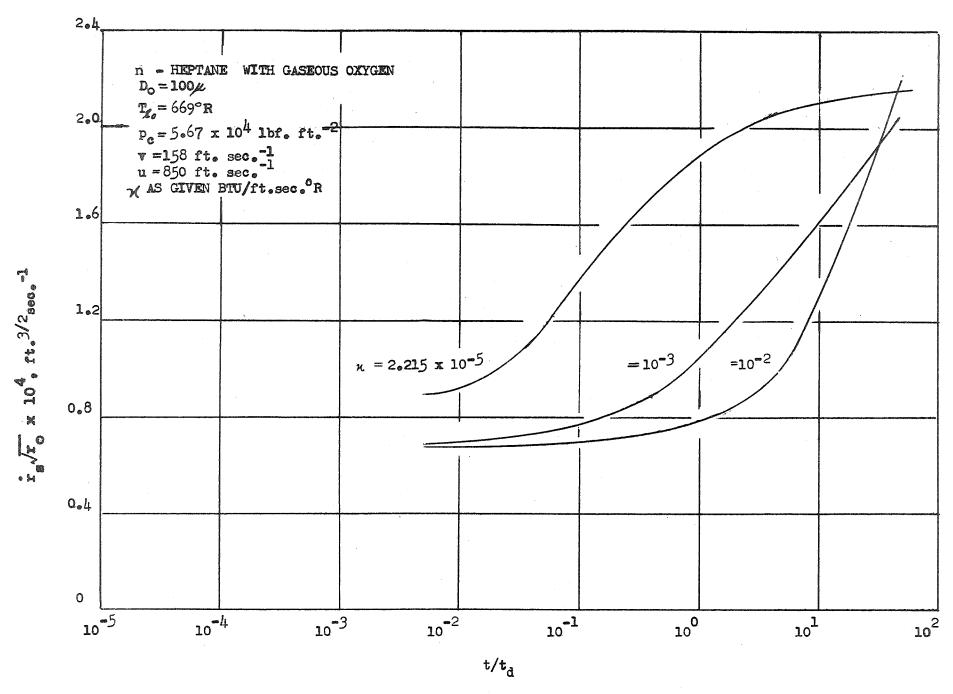


FIG. 8d. SURFACE REGRESSION RATE VS. TIME FOR VARIATION IN PROPELLANT THERMAL CONDUCTIVITY, n-HEPTANE WITH GASEOUS OXYGEN

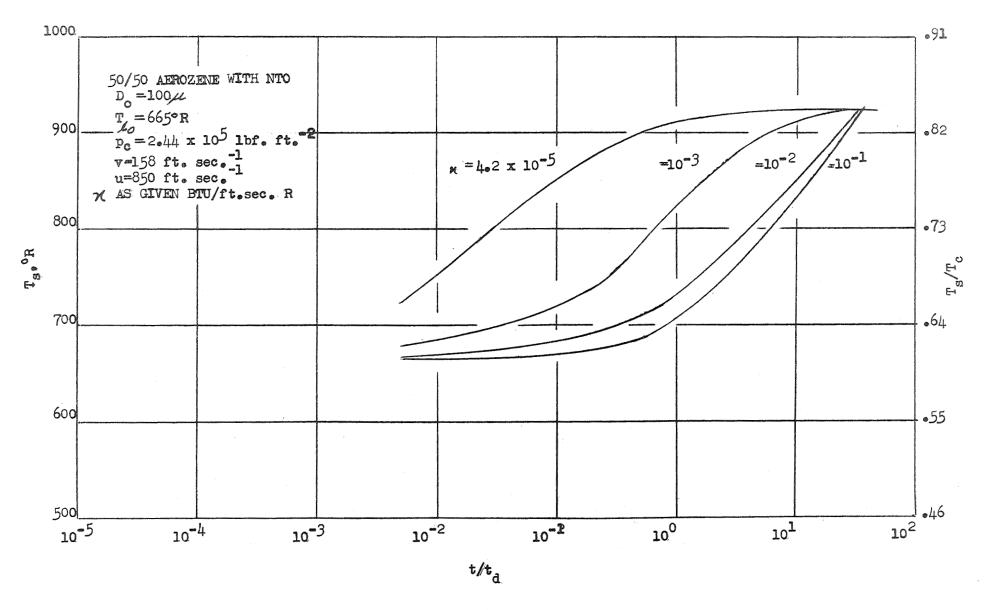


FIG. 9a. SURFACE TEMPERATURE VS. TIME, 50/50 AEROZINE WITH NTO

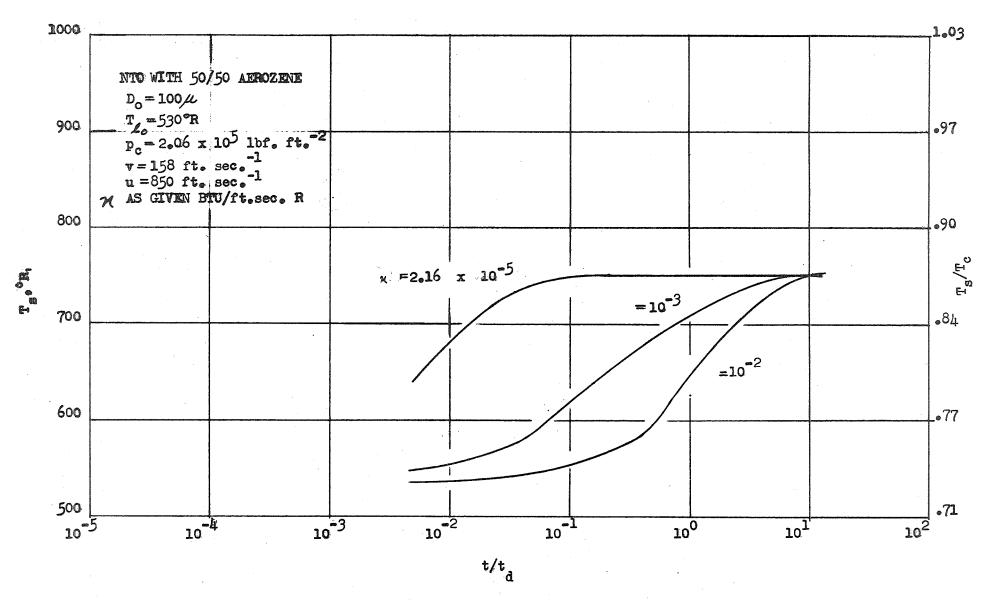


FIG. 9b. SURFACE TEMPERATURE VS. TIME, NTO WITH 50/50 AEROZINE

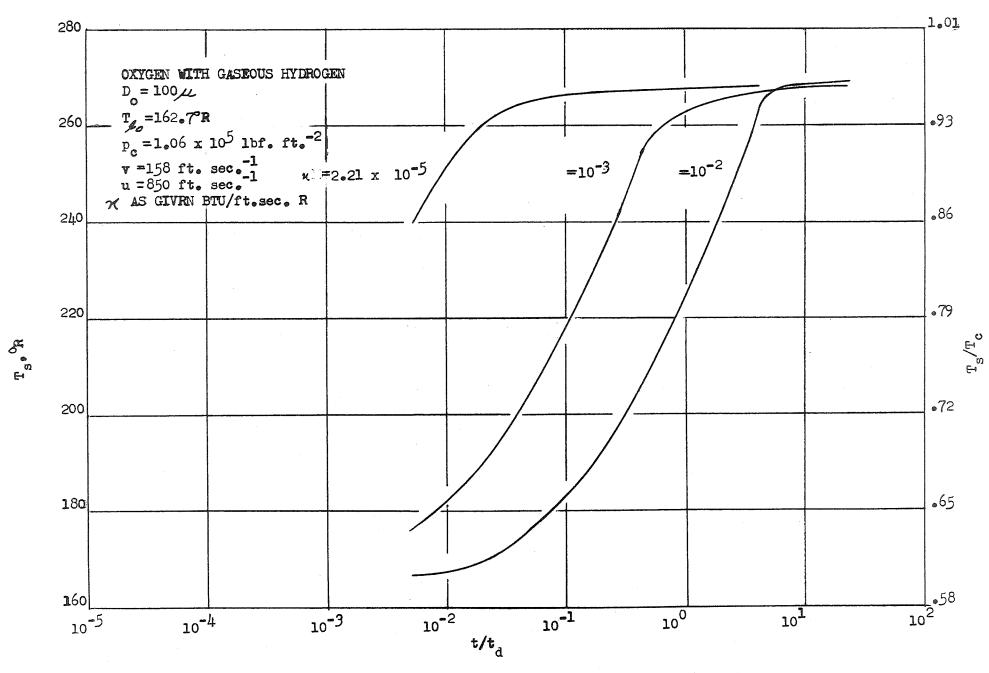


FIG. 9c. SURFACE TEMPERATURE VS. TIME, OXYGEN WITH GASEOUS HYDROGEN

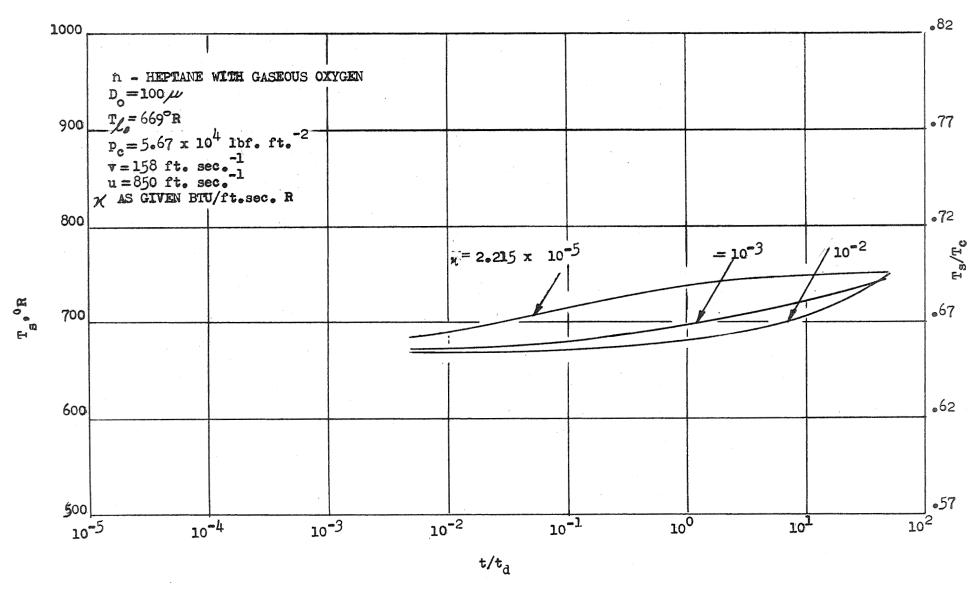


FIG. 9d. SURFACE TEMPERATURE VS. TIME, n-HEPTANE WITH GASEOUS OXYGEN

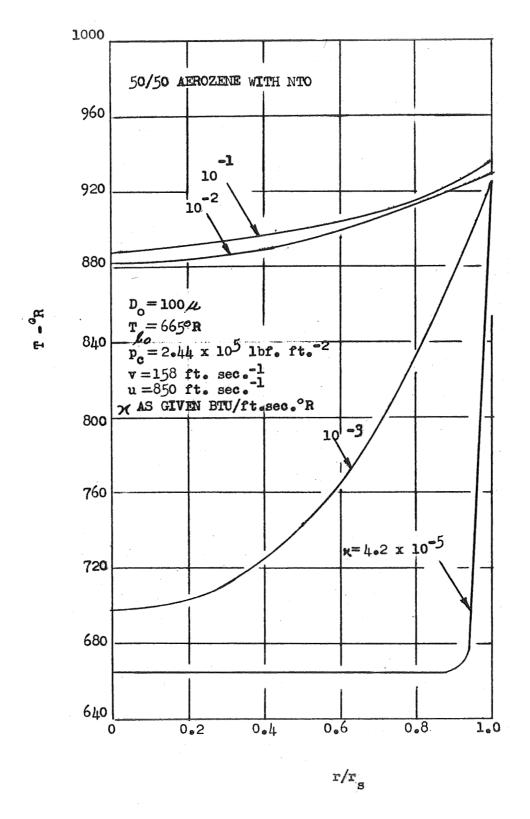


FIG. 10a. DROPLET TEMPERATURE DISTRIBUTION, AT TEMPERCENT DROPLET MASS EVAPORATION, 50/50 AEROZINE WITH NTO

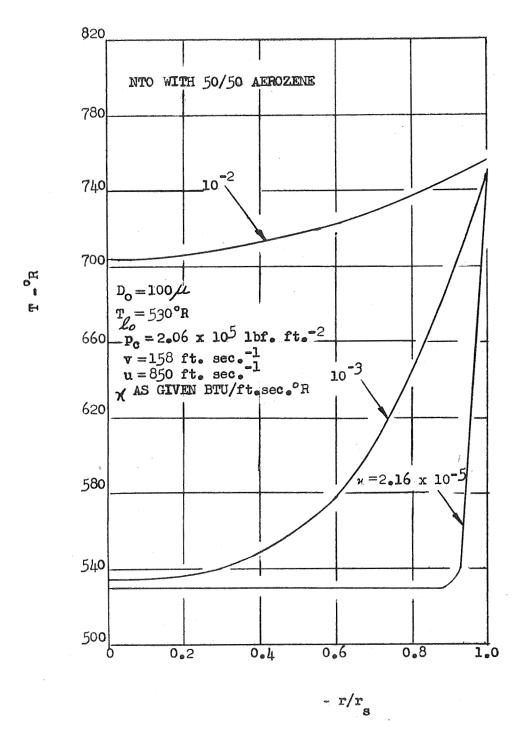


FIG. 10b. DROPLET TEMPERATURE DISTRIBUTION, AT TEM PERCENT DROPLET MASS EVAPORATION, NOO WITH 50/50 AEROZINE

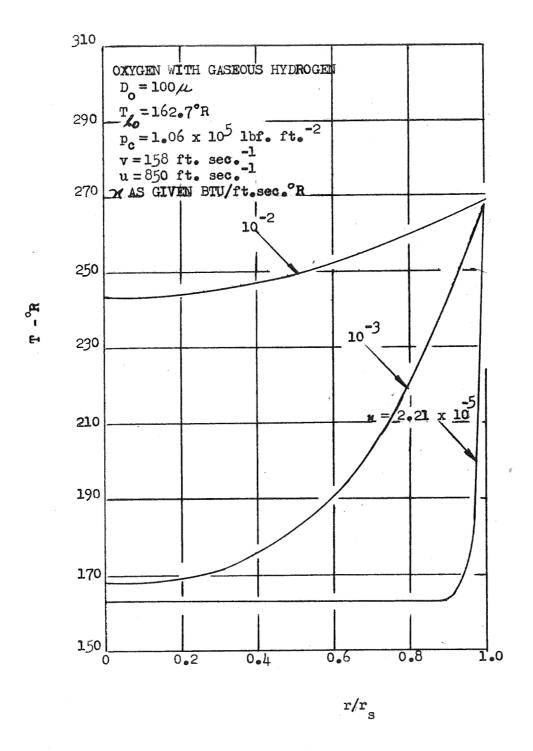


FIG. 10c. DROPLET TEMPERATURE DISTRIBUTION, ATTEM PERCENT DROPLET MASS EVAPORATION, OXYGEN WITH GASEOUS HYDROGEN

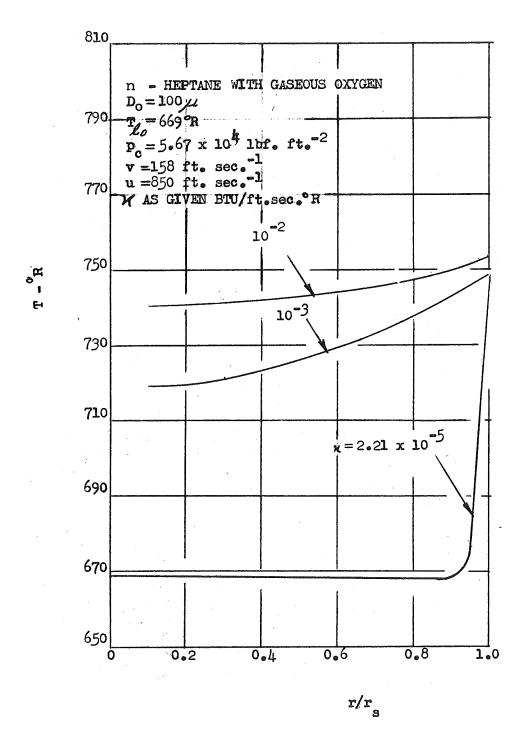


FIG. 10d. DROPLET TEMPERATURE DISTRIBUTION, AT TEN PERCENT DROPLET MASS EVAPORATION, n-HEPTANE WITH GASEOUS OXYGEN

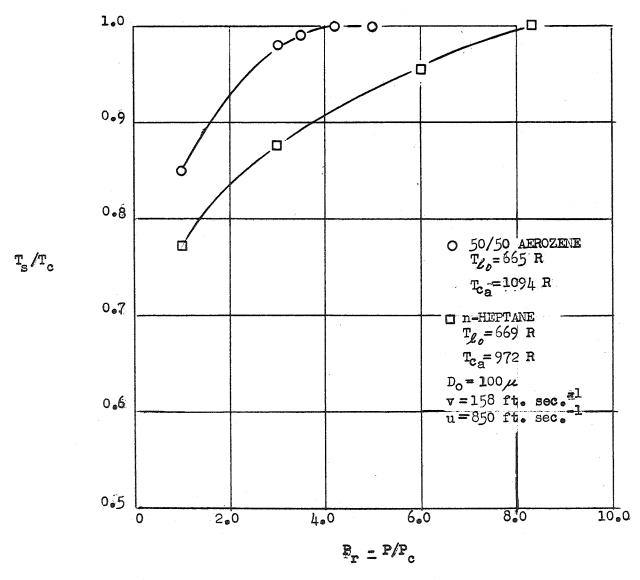


FIG. 11a. DROPLET SURFACE TEMPERATURE VS. ENVIRONMENTAL GAS PRESSURE, n-HEPTANE WITH GASEOUS OXYGEN, 50/50 AEROZINE WITH NTO

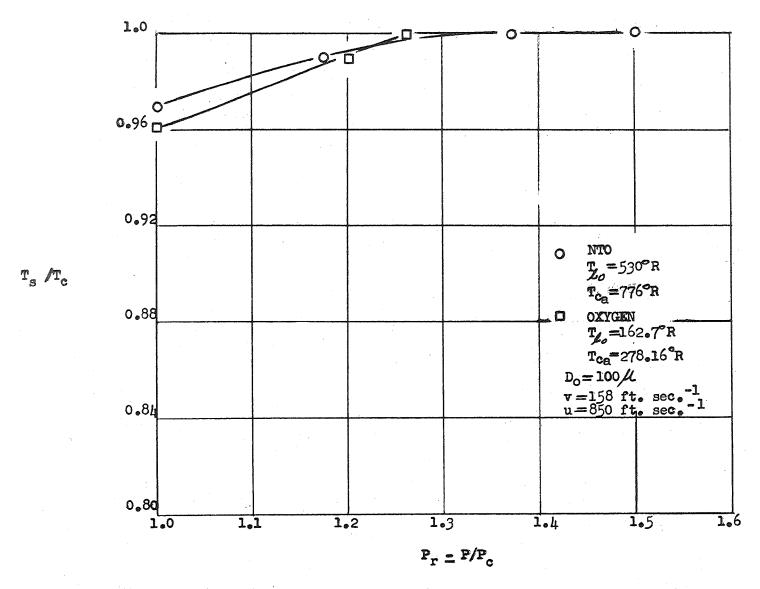


FIG. 11b. DROPLET SURFACE TEMPERATURE VS. ENVIRONMENTAL GAS PRESSURE, NTO WITH 50/50 AEROZINE, OXYGEN WITH GASEOUS HYDROGEN

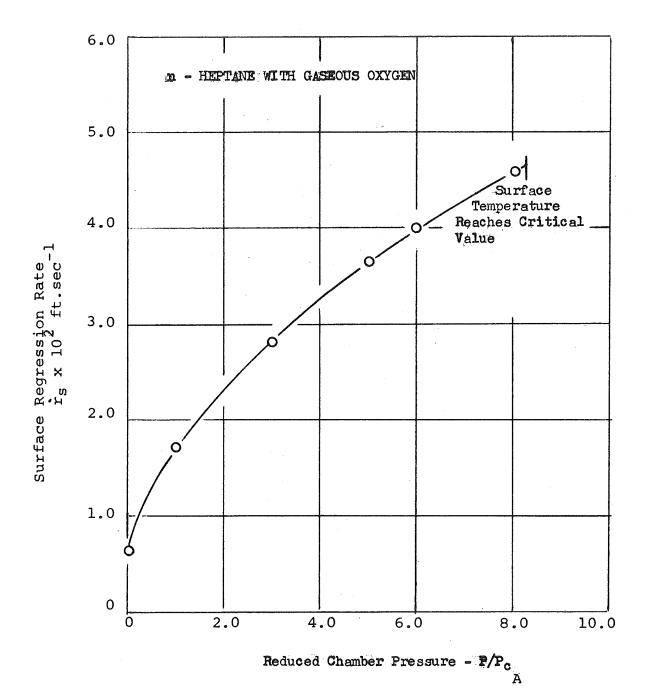


FIG. 12 SURFACE REGRESSION RATE AT SUPERCRITICAL PRESSURES FOR n-HEPTANE

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